



Arab Academy

for Science , Technology and Maritime Transport



The International Maritime Transport
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“MARLOG 13”

**Towards _____
Smart Green Blue
Infrastructure**

3-5 March 2024 - Alexandria, Egypt





Eng. Ahmed Abdallah

Elevating Wind Energy Harvesting with J-shaped Blades: A CFD-driven Analysis of H-Darrieus Vertical Axis Wind Turbines

Ahmed Abdallah , Micheal A. William and Iham F. Zidane 3





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2. Research Objectives
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 - Numerical setup
4. VERIFICATION AND VALIDATION
5. J-BLADE CONFIGURATION
6. RESULTS AND DISCUSSION
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 - Standard Operational Conditions
7. Conclusions

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Background Motivation

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- Energy shortages and environmental deterioration drive renewable energy demand. Due to technical advances, wind energy has a low environmental effect and low operational costs [\[1\]](#).
- Wind turbines have a either horizontal or vertical axis. Large onshore and offshore projects require HAWTs, whereas suburban areas prefer VAWTs, for which low wind-speed has a large impact on the turbine's performance.



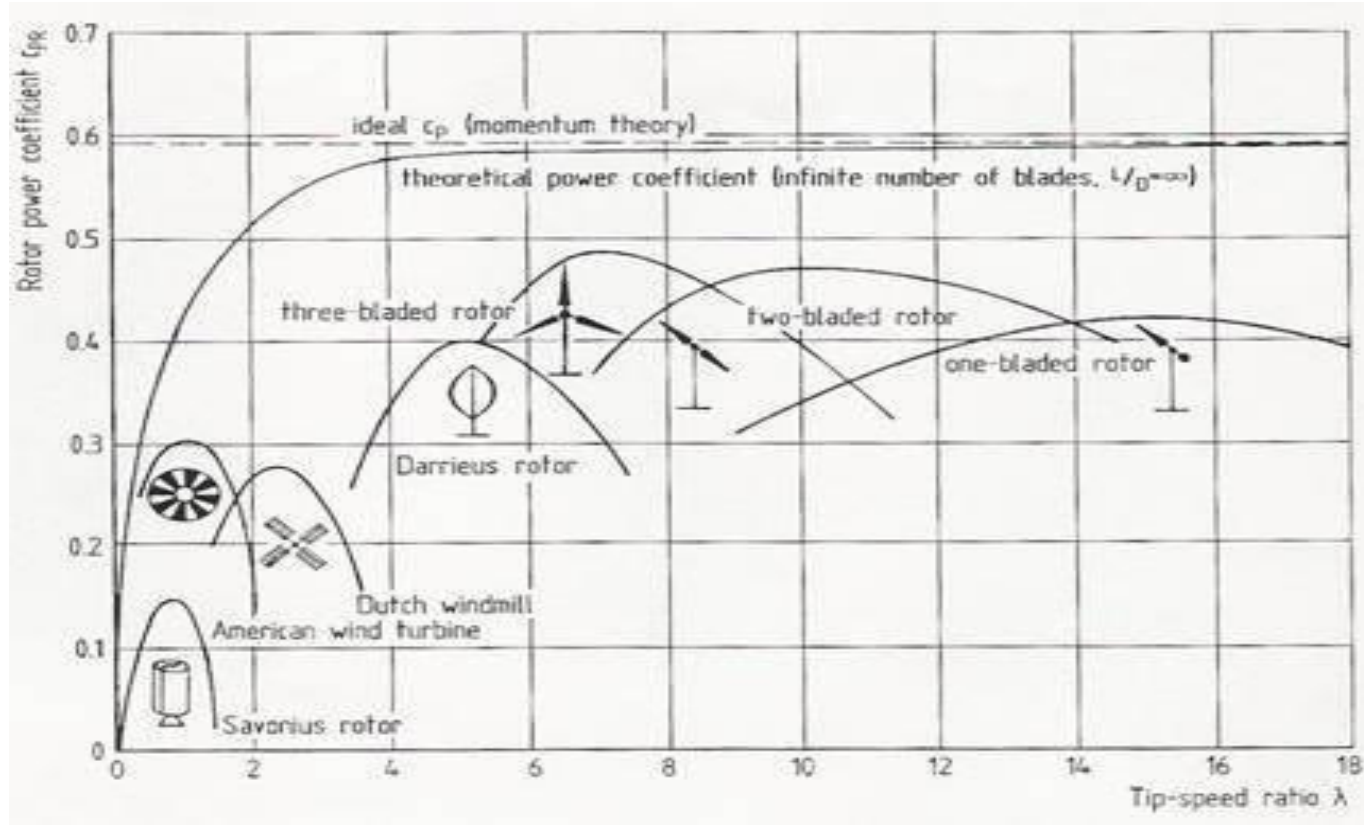


Figure 1. Comparison of turbines performance VS TSR



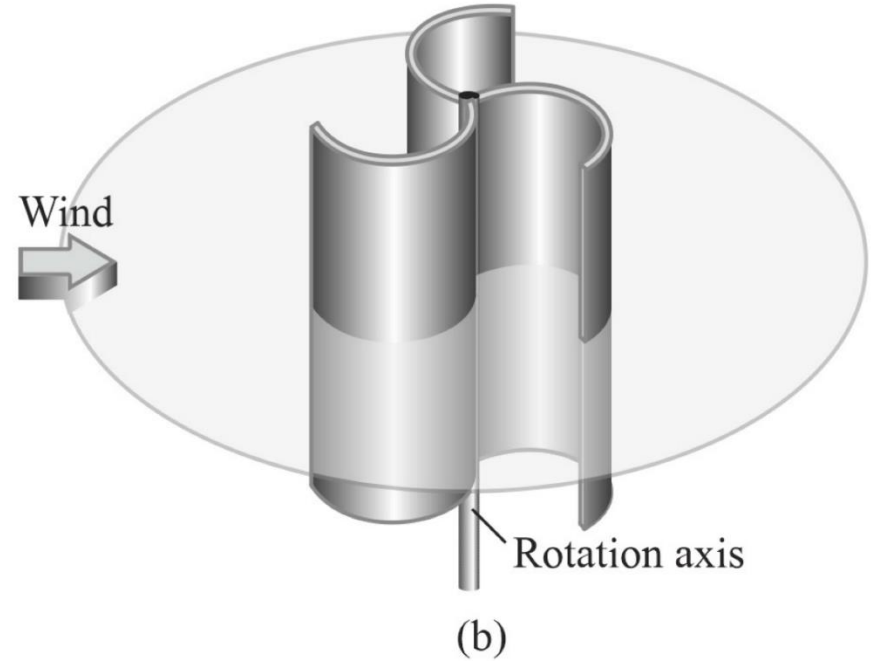
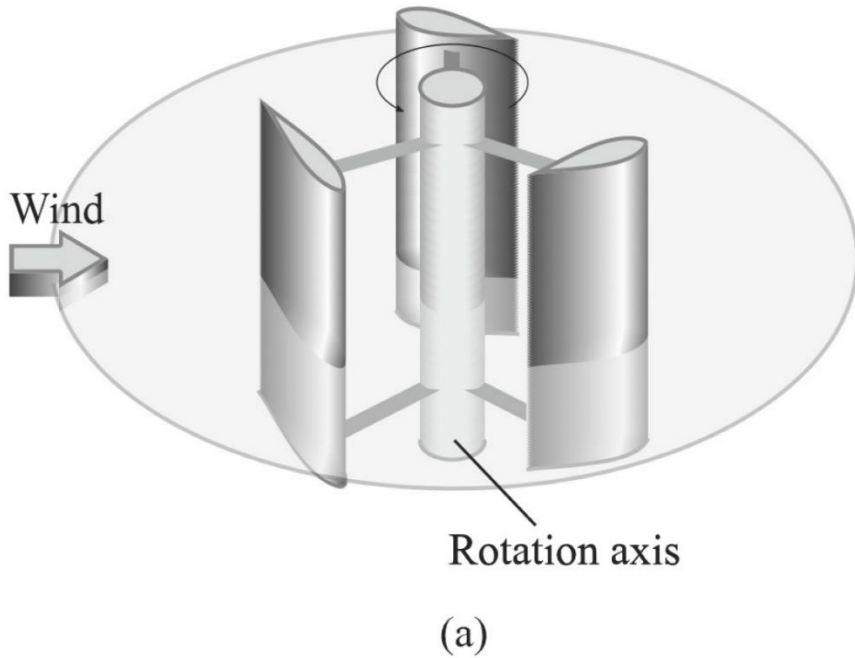


Figure 2. (a) Darrius Turbine; (b) Savanous Turbine

However, other design strategies have been investigated, such as:

- the use of helical blades, or the addition of flaps or vortex generators to the blades [15].
- the use of augmentation technologies on vertical DAWT [4,16].
- Slotted blades have also been investigated recently [17,18].
- Upstream Deflector blades have also been investigated recently [25].
- One innovative idea was through opening the blade trailing edge, and forming what is called a J-shaped blade [19,20].

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WSU0015



NACA 0018



SAND 0021



TWT11215-1



NACA632-015



DU 06-W-200



LS-0417



S1210



NTU-20-V



GOE 420

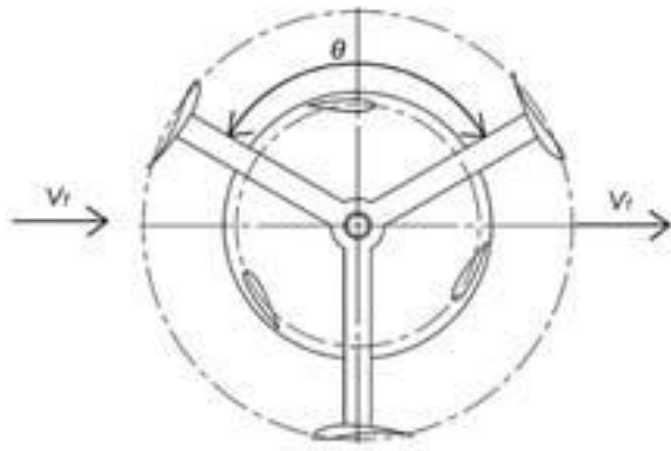


NACA 4415

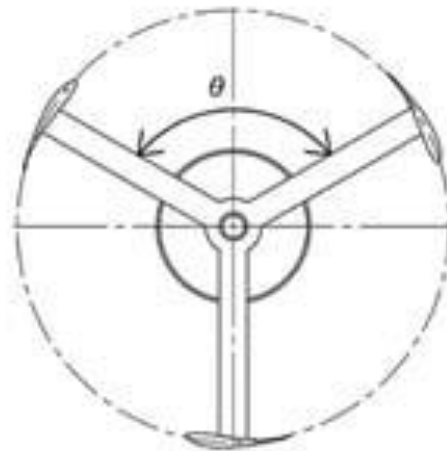


NLF-0416

Figure 3. Airfoil Characteristics



(a)



(b)



(c)

Figure 4. (a) Concave-In; (b) Concave-Out; (c) Turbine with cambered blades.



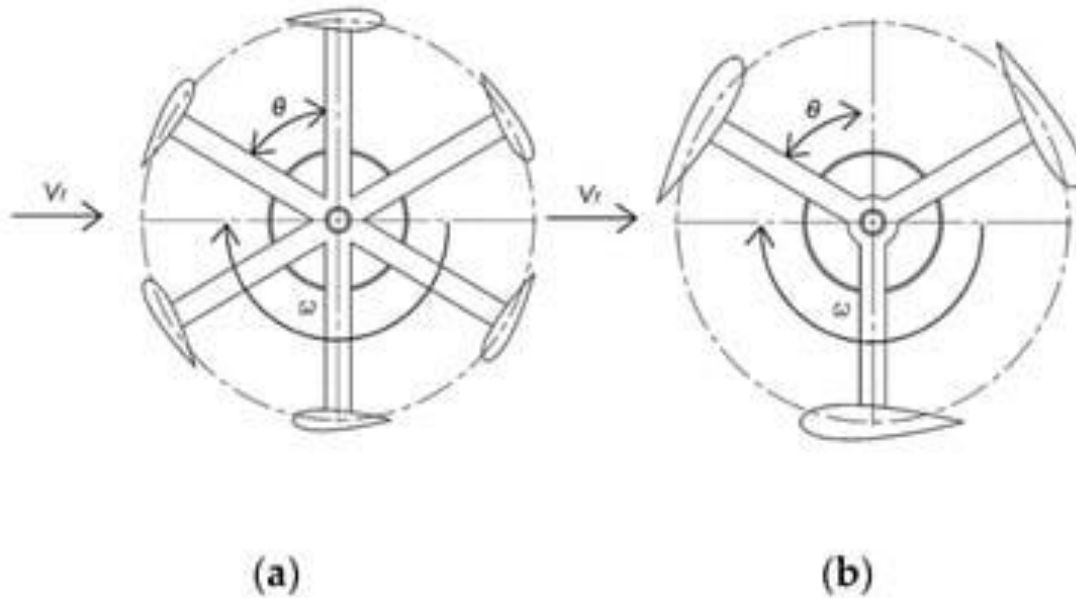


Figure 5. (a) Increased number of blades (b) Increased chord (c) High solidity turbine

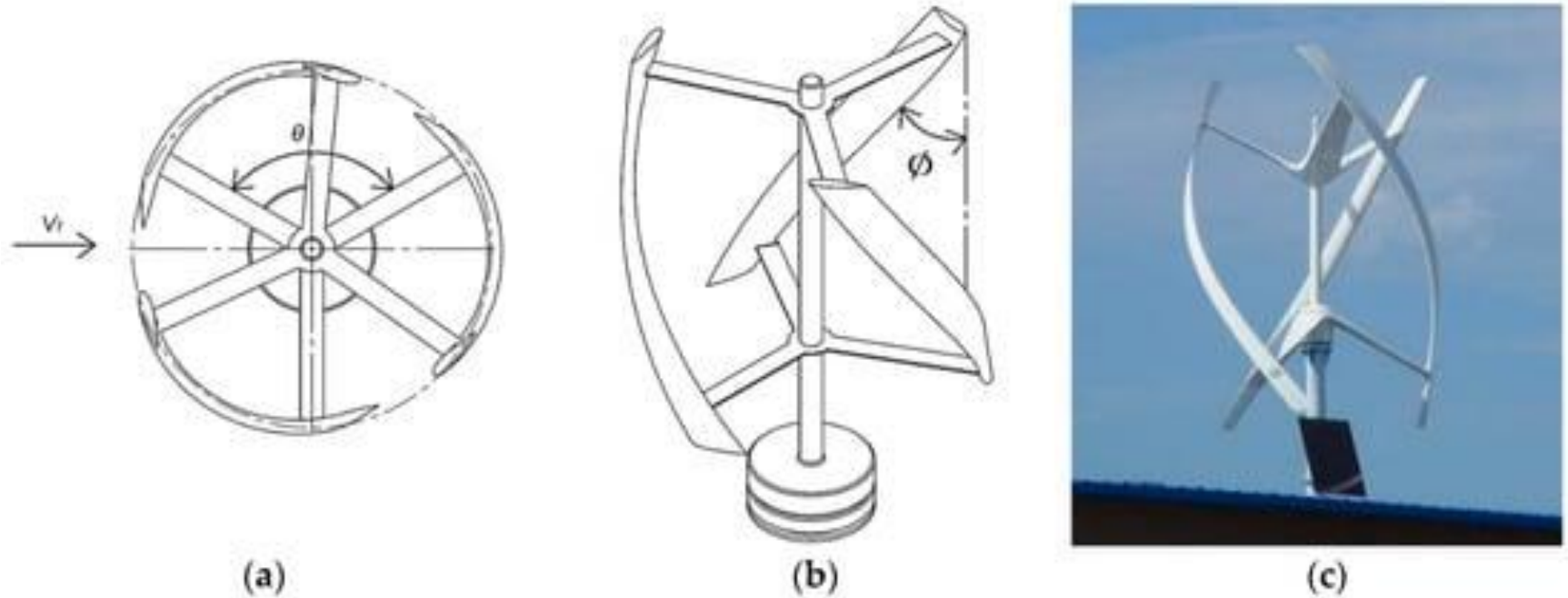


Figure 6. (a) Top view (b) Front view of helical rotor (c) Commercial helical turbine.

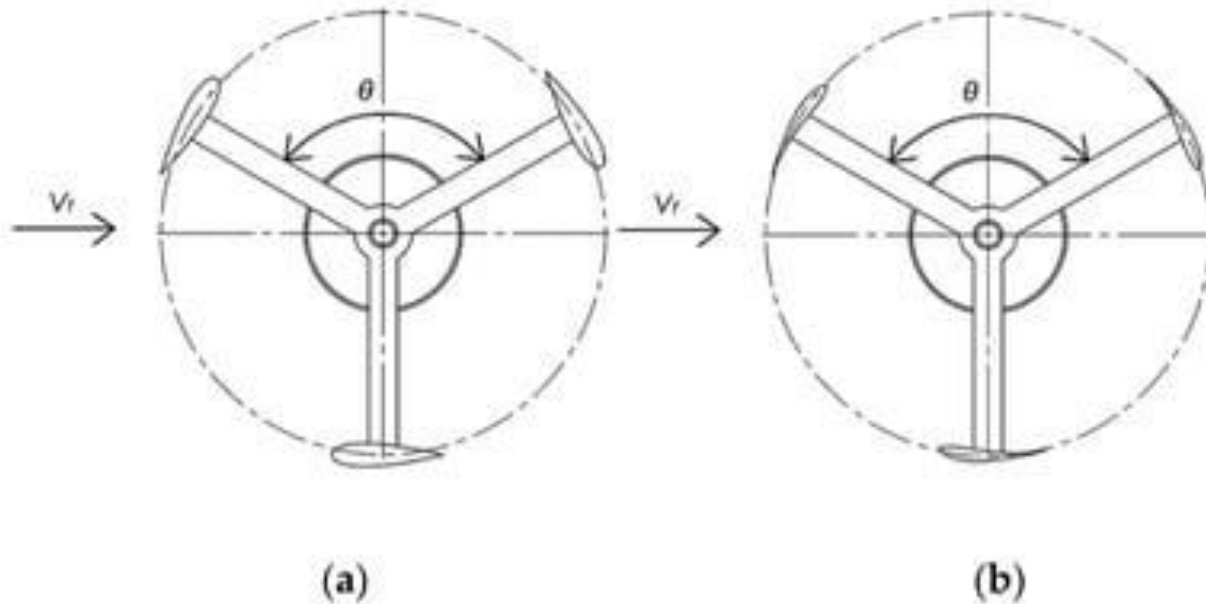
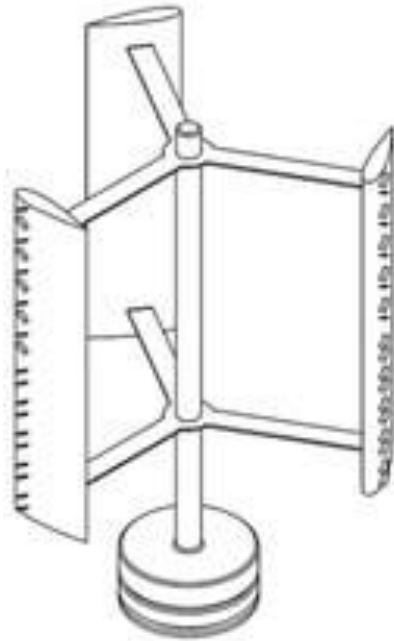
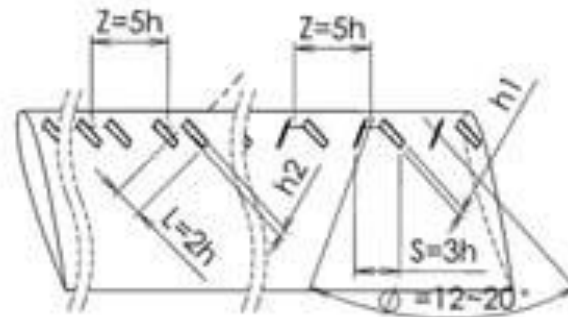


Figure 7. (a) H-Rotor with NACA 0021 (thick) airfoil (b) H-Rotor with S1210 (thin) airfoil (c) Commercial turbine with thicker blade.





(a)



(b)



(c)

Figure 8. (a) H-Rotor with VG; (b) Typical VG details; (c) DAF INDAL turbine with VG.

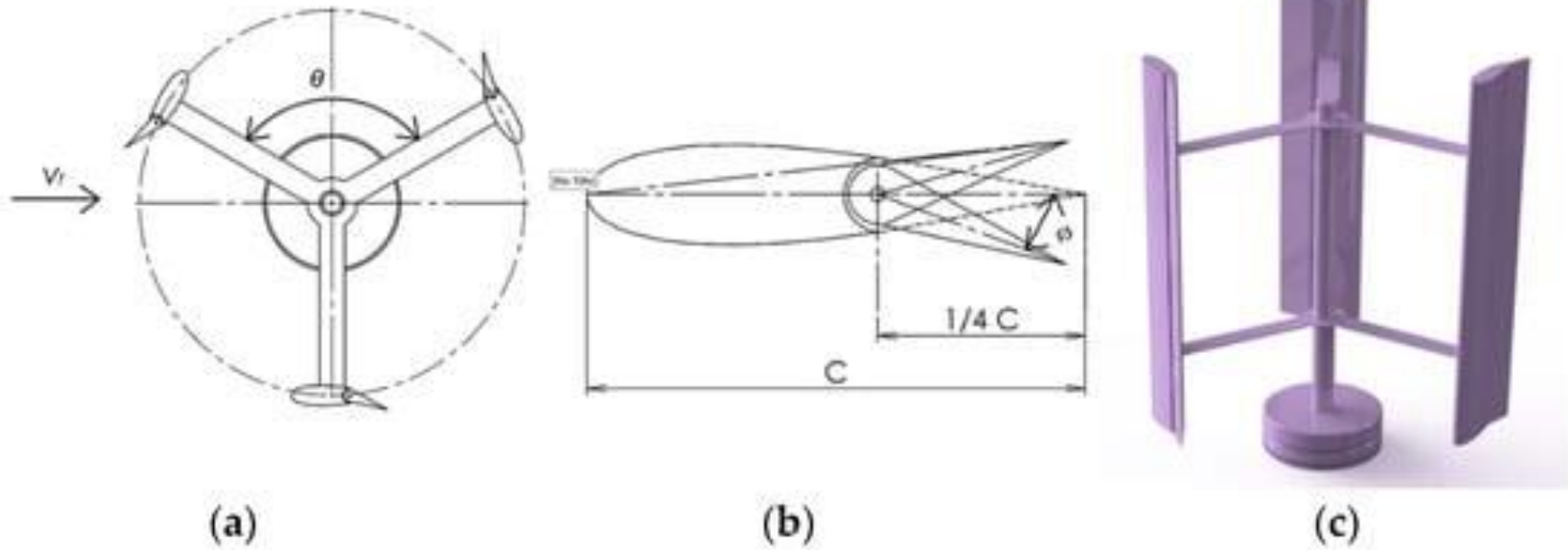


Figure 9. (a) H-rotor with a trailing edge flap (b) Details of an airfoil with a trailing edge flap (c) Illustration of a H-rotor with trailing edge flaps.

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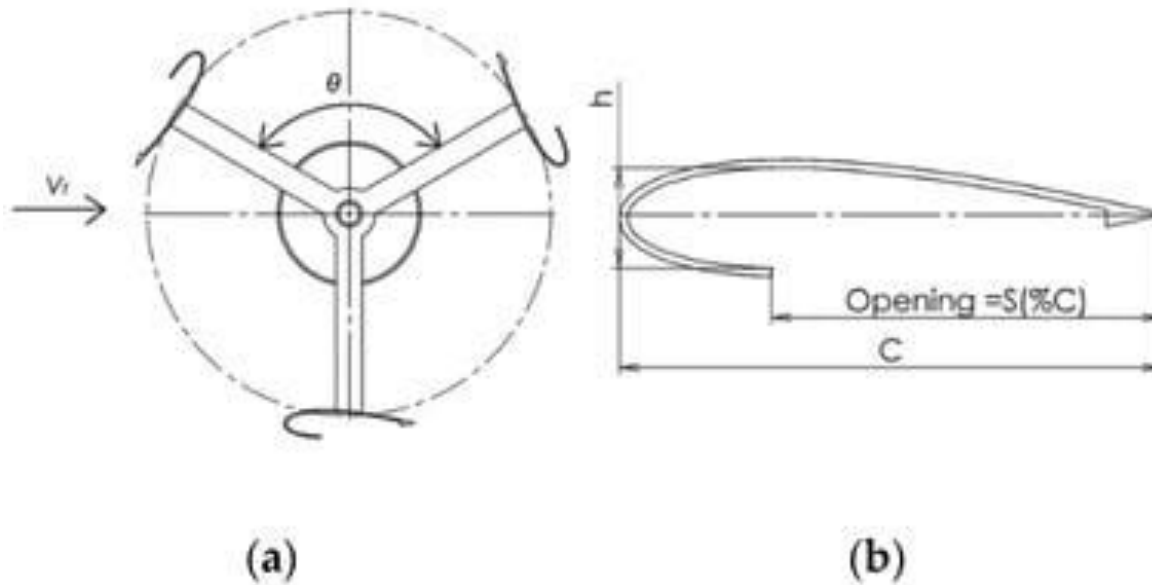


Figure 10. (a) H-Rotor with J-profile airfoil (b) Details of J-profile airfoil (c) Commercial turbine with J-profiled blade.

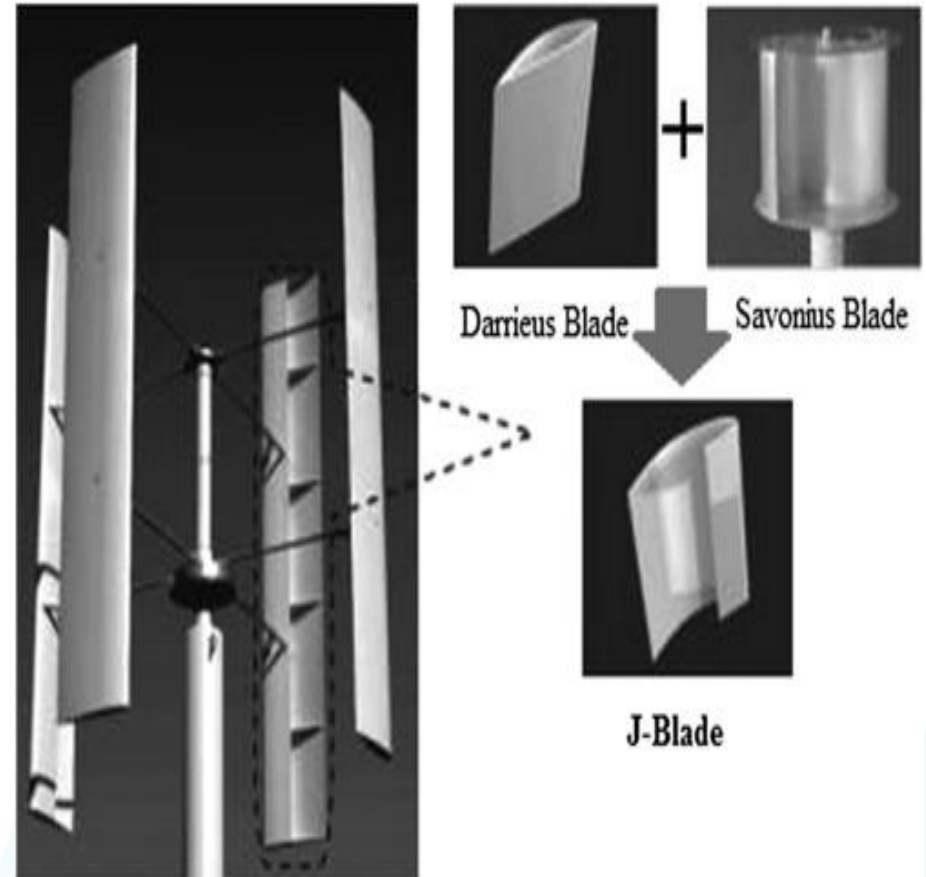


Table 1. Comparison of strategies, their merits, and demerits.

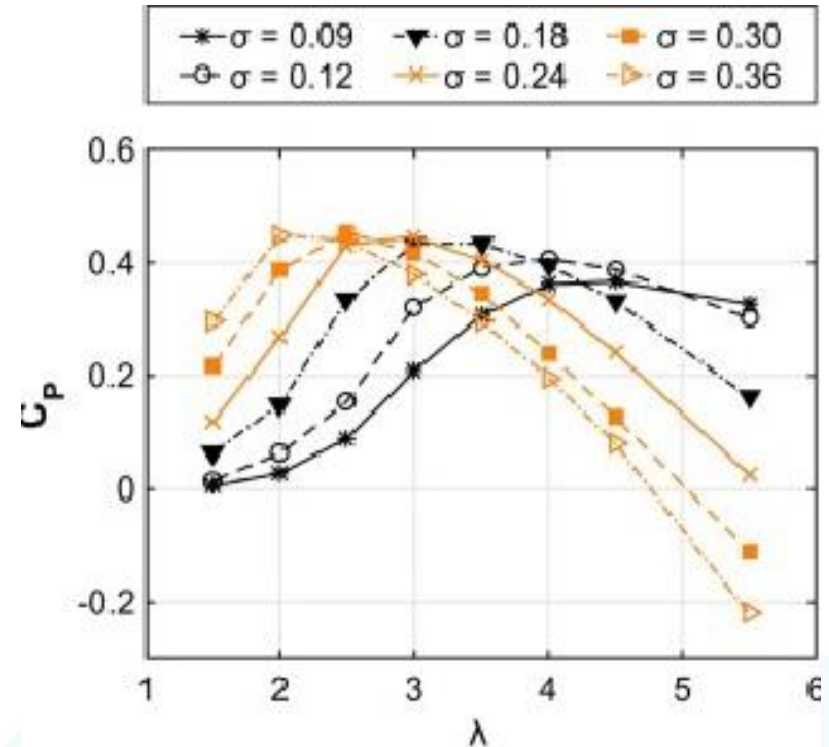
Strategy	Merits	Demerits
Airfoil characteristics	Less complex, cost-effective, better optimization of dynamic stall, blade tip loss. Increased structural rigidity	Minimal improvement in starting performance and overall power coefficient
Cambered airfoil	Better starting capability delayed stall, less sensitive to surface roughness, high pitching moment	Increased drag on the down half, reduction in power coefficient
Solidity	Increased starting torque, low centrifugal forces due to low rpm	Large AoA, large drive train size and additional cost due to an increased number of blades
Helical blades	Improved aesthetics, smooth torque pulsation leading to reduced vibration	High manufacturing cost of blades, low torque for every azimuthal position
Blade thickness	Better performance at low Re delayed stall and structurally sound	Increased profile drag at high Re and noise
Vortex generators	Improved performance at low Re, extended dynamic stall and marginal improvement in starting torque	Increased drag at high TSR and increases noise due to vortex shedding
Gurney flaps	Delayed dynamic stall with an increase in starting torque compared to a conventional airfoil	Vibration after stall and noise due to vortex shedding
Trailing edge flaps	Better performance in high and low TSR, able to regulate the rotor rpm aerodynamically	Power has to be expended to operate flaps, requires a sophisticated control system
J-Blade	Excellent startup torque, able to sustain low wind speed rotation, ease of blade manufacturing	Degraded performance at high TSR due to high form drag at high Re and increased fatigue failure of blades



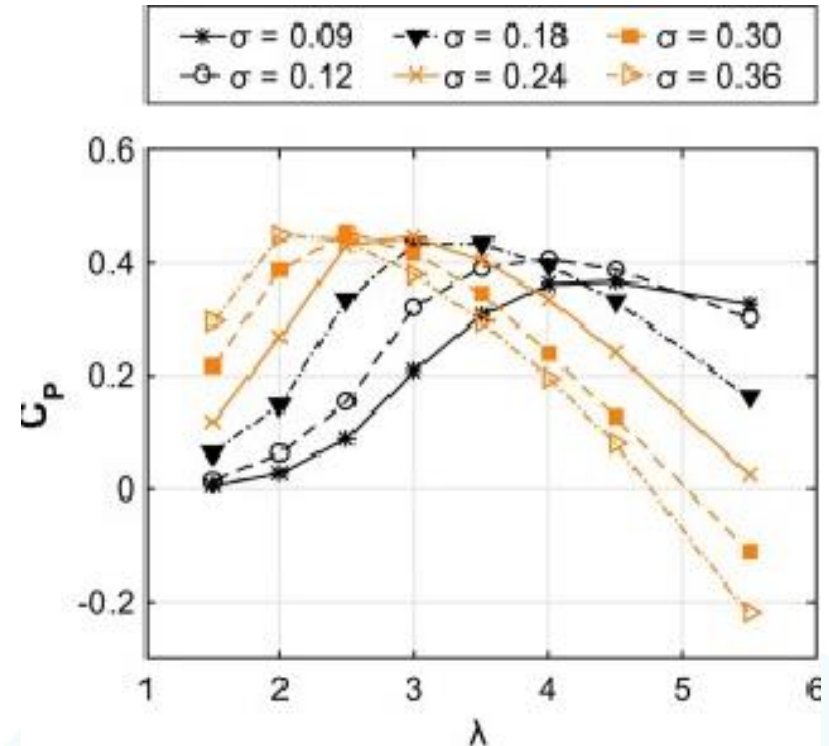
- Numerical studies on J-shaped airfoils [19,20] have demonstrated that the self-starting ability of a VAWTs could be improved using an opening on the airfoil. Research indeed showed an almost linear starting torque enhancement with a cut in the outer part of the blade [19].
- Other authors had different views, they claimed that a J-shaped blade design does not provide any performance enhancement [22], and they did not recommend their use for Darrieus-type VAWTs.



- Further research was performed here to evaluate the effect of the solidity and λ on the performance of Darrieus VAWTs. Two main regions are usually considered for the operational regions of Darrieus turbines, either operating in the high TSR region ($\lambda = 5$ or $\lambda = 6$) or operating in the low TSR region ($\lambda = 1.5$) [23].



- For high TSR values, the wind flow is mainly attached to the airfoil, due to the low angles of attack. However, when the TSR is in the low region, the angle of attack of the flow on the airfoil increases, leading to dynamic stall effects that highly influence the turbine performance [23]
- **Thus, high solidity (typically around 0.3–0.5) and low TSR seem favorable for the J-shaped design.**



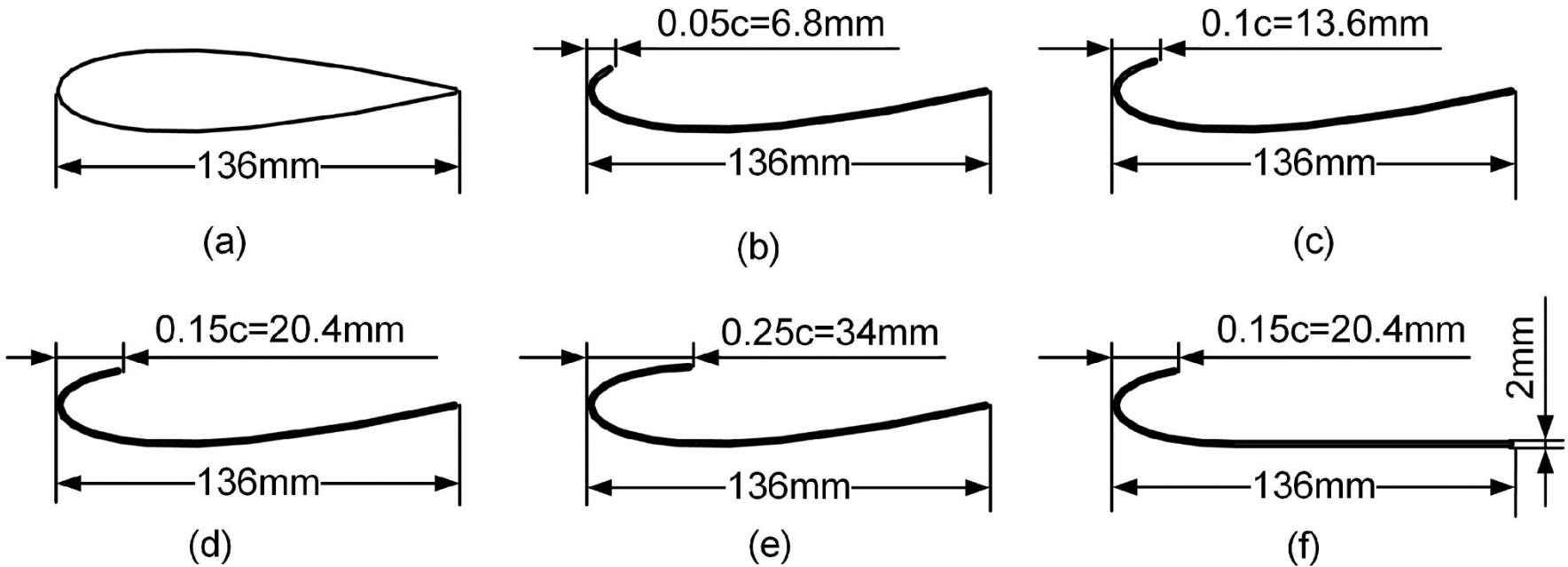


Figure 11. J-blade with different openings





Research Objectives

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- This research aims to create an **optimized J-shaped blade** to enhance the Darrieus turbine's performance by encouraging resilient, sustainable infrastructure that is built and run to have the least negative effects on the environment that maintains the Darrieus turbine's performance while improving starting torque, utilizing a NACA0015 airfoil as a model.
- The earlier research on J-shapes [5-7] and the more recent study [10] concentrated **on blades with a hollow and hair-like form**; however, they did not investigate the inner shape of the construction. Thus, this study evaluates the performance of J-shaped blades with an **interior-filled construction**.

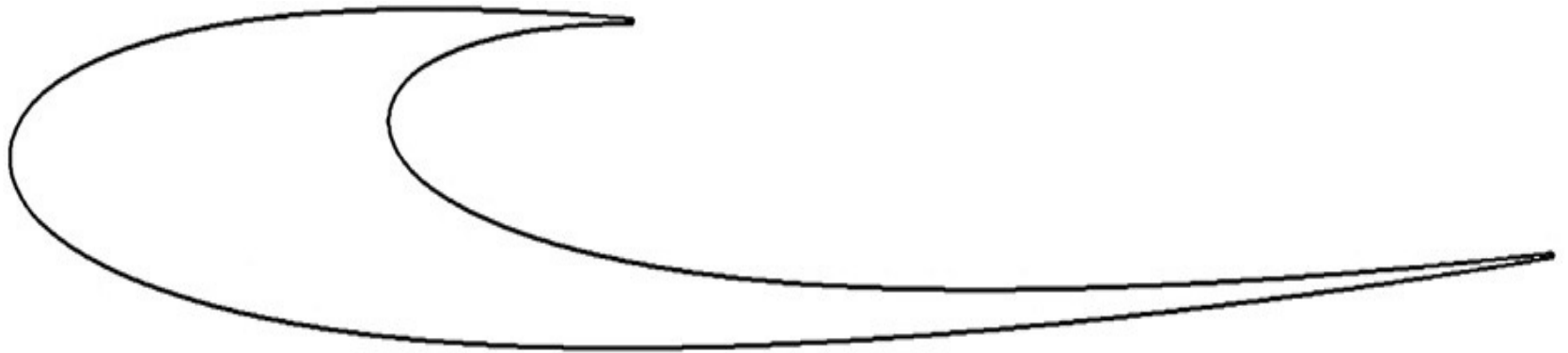


Figure 12. J-blade airfoil design

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Problem description and setup

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- This study leverages the **Finite Volume Fluent Solver** to meticulously investigate the impact of J-shaped configurations on the aerodynamic performance of airfoils for vertical axis wind turbines, employing the **Unsteady Reynolds Averaged Navier Stokes (URANS)** governing equations. The research endeavors to contribute to the understanding of these configurations by presenting a comprehensive analysis through **a turbulent, incompressible, and two-dimensional flow model.**

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H-Darrieus wind turbine geometry features

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Characteristics	Details
No. of blades	3
Rated power	3.5 kW
Chord length	0.4 m
Turbine radius	1.25 m
Blade profile	NACA0015
Solidity	0.48
Wind speed	10 m/s
Reynolds no.	416,254

Table 2. Main geometrical rotor characteristics



Wind turbine governing equations

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- The validity of the model in this study has been examined by a comparison of the coefficient of power (C_p) vs. tip speed ratio (λ) curve with findings reported in previous studies.
- $C_p = C_m (\omega L)/u$ (1)
- $C_p = C_m (\omega R)/u$ (2)
- $\lambda = (\omega R)/u$ (3)
- $C_p = C_m \lambda$ (4)

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Gromeka acceleration vector (m/s^2) is the rate of change of angular momentum [16]. According to [17], The total external torque acting on a system of particles equals the rate of change of the system's total angular momentum.

The torque created on the turbine is evaluated by measuring the Gromeka acceleration vector.

For a 2D flow, the velocity (u) may be defined as

$$\vec{u} = \begin{pmatrix} u \\ v \\ 0 \end{pmatrix} \quad (5)$$

And we can determine vorticity using equation (6). according to topics

$$\vec{\xi} = \nabla \times \vec{u} = \begin{pmatrix} \xi_x \\ \xi_y \\ \xi_z \end{pmatrix} \quad (6)$$



For a 2D flow,

$$\vec{\xi} = \nabla \times \vec{u} = \begin{pmatrix} 0 \\ 0 \\ \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \end{pmatrix} \quad (7)$$

To extract the Gromeka acceleration vector $\vec{G} = \vec{\xi} \times \vec{u}$
(8)

For a 2D flow, the Gromeka acceleration vector may be defined as

$$\vec{G} = \begin{pmatrix} v\xi_z \\ -u\xi_z \\ 0 \end{pmatrix} = \begin{pmatrix} v\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right) \\ -uv\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right) \\ 0 \end{pmatrix} \quad (9)$$

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To determine the Gromeka acceleration vector magnitude, this equation is utilized.

$$|\vec{G}| = \sqrt{\left(v \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)\right)^2 + \left(-u \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)\right)^2} \quad (10)$$

$$= \sqrt{\left(\left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)\right)^2 \times (u^2 + v^2)} \quad (11)$$

$$= \sqrt{(|\vec{\xi}|)^2 \times (|\vec{u}|)^2} \quad (12)$$

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Numerical details and solver setup

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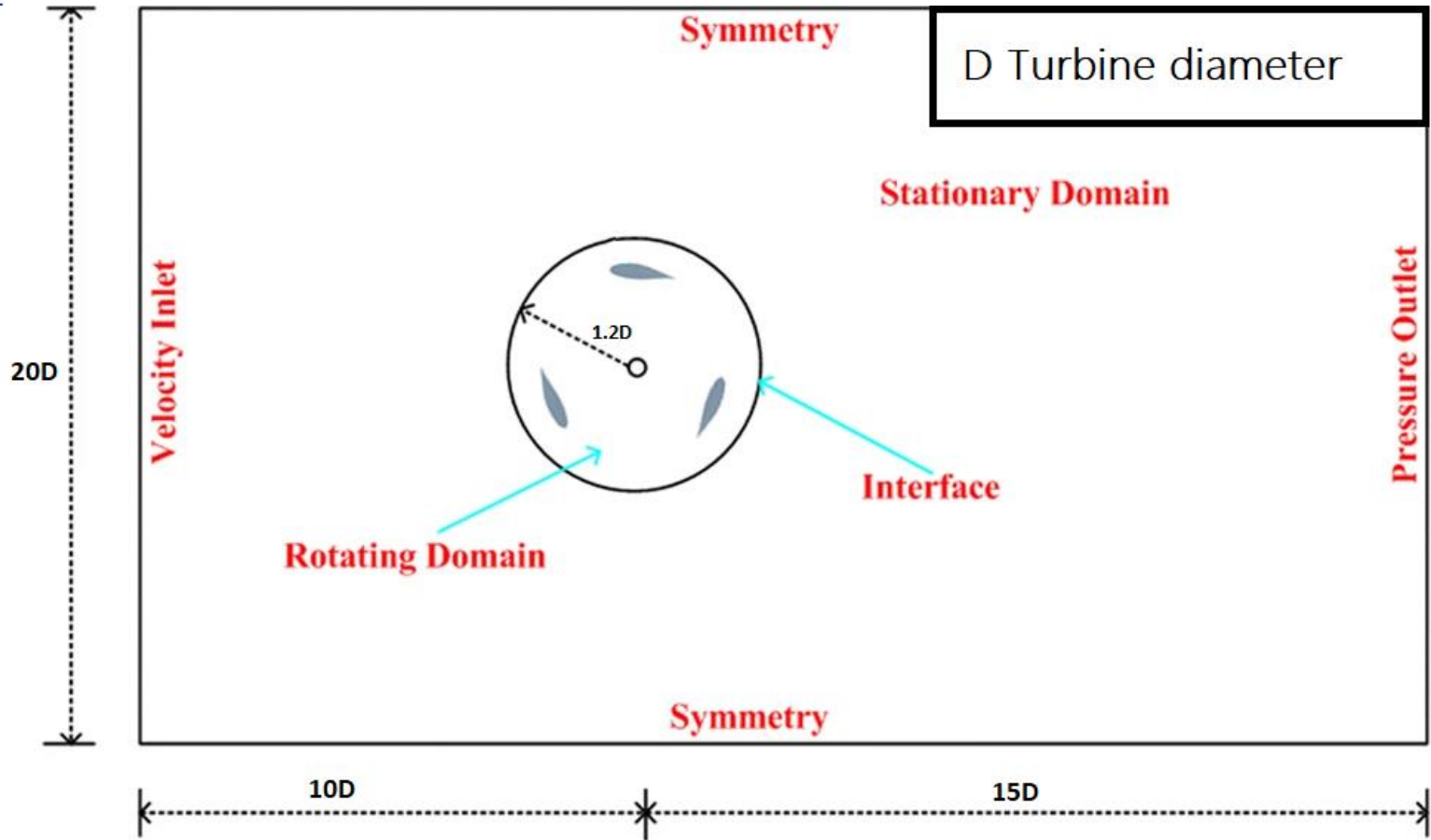
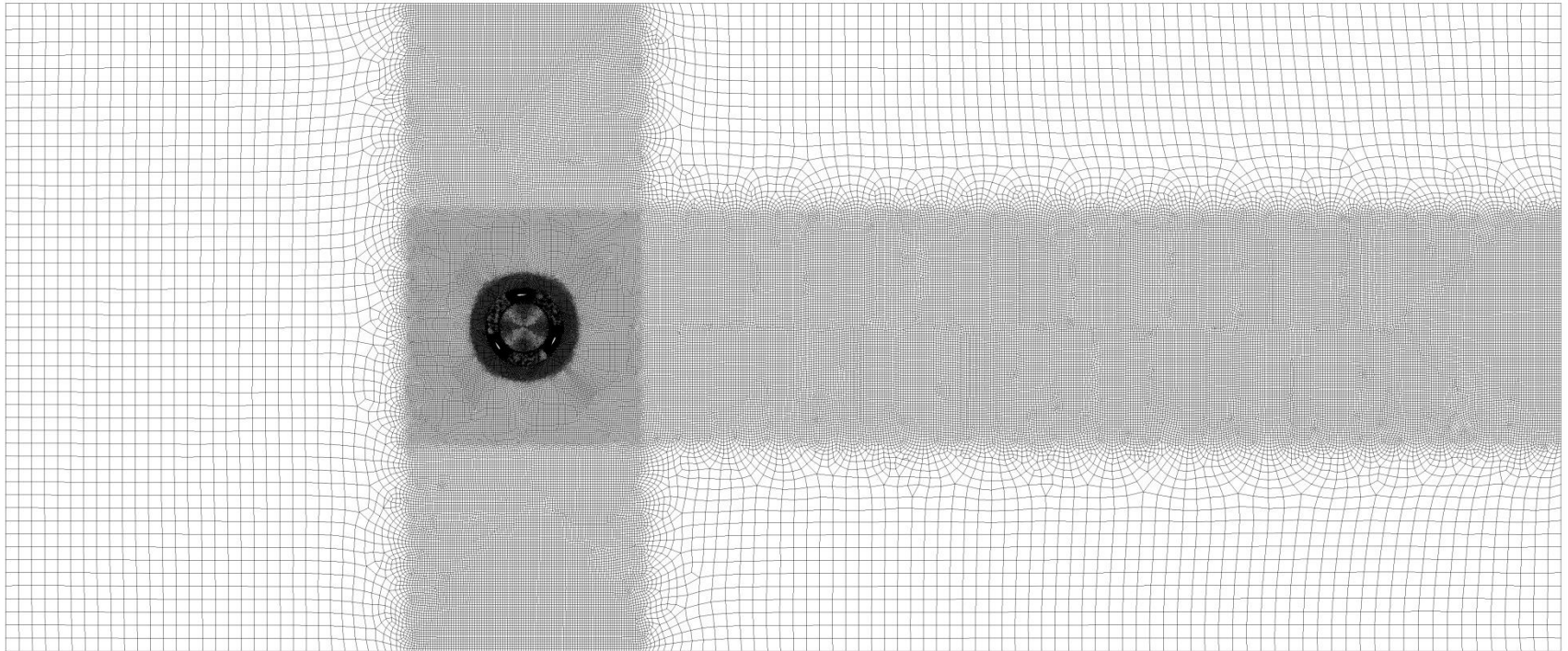


Figure 13. Computational domain



0 10 (m)

Figure 14. Grid distribution around NACA 0015

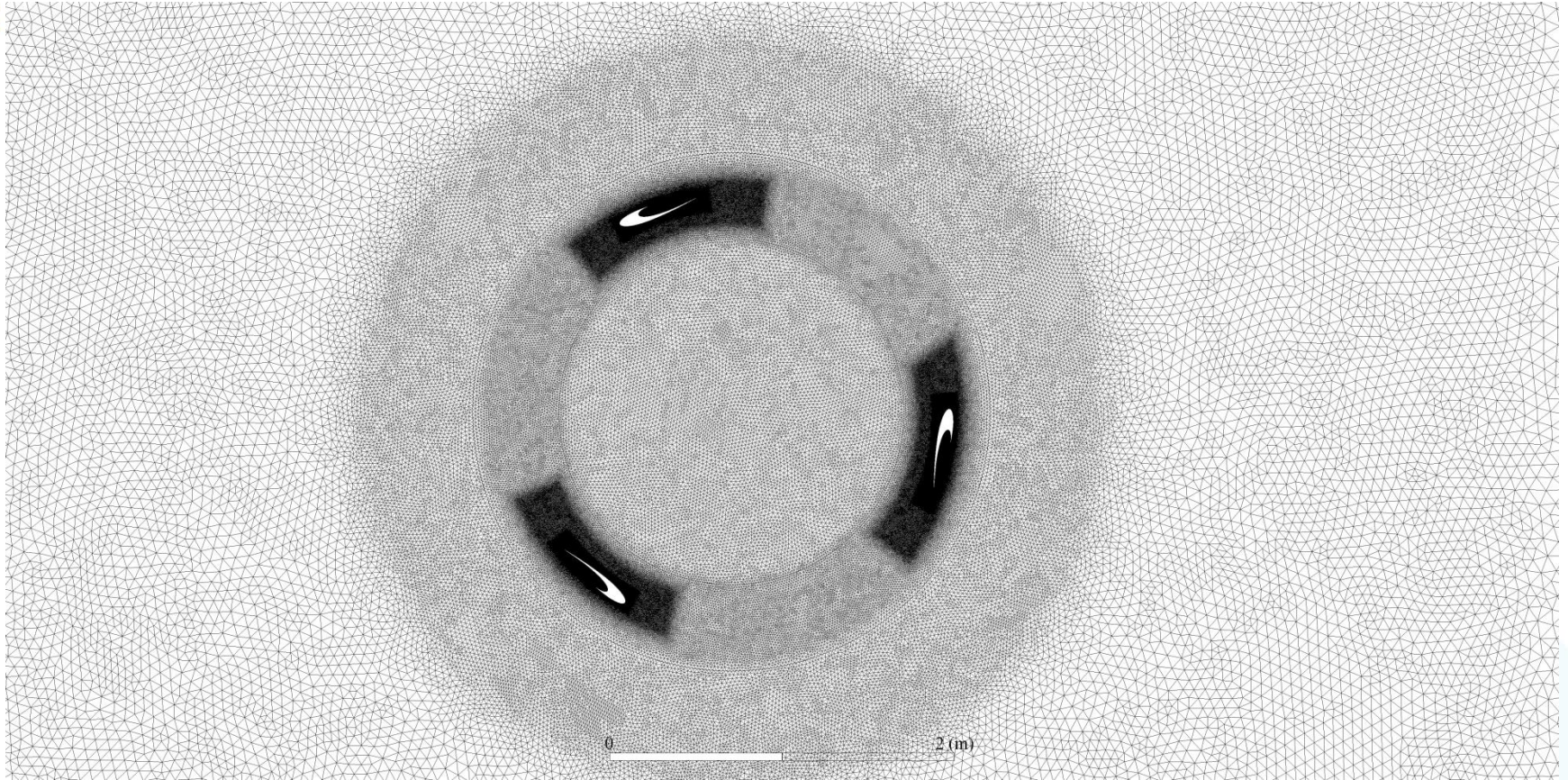


Figure 15. Grid distribution around NACA 0015

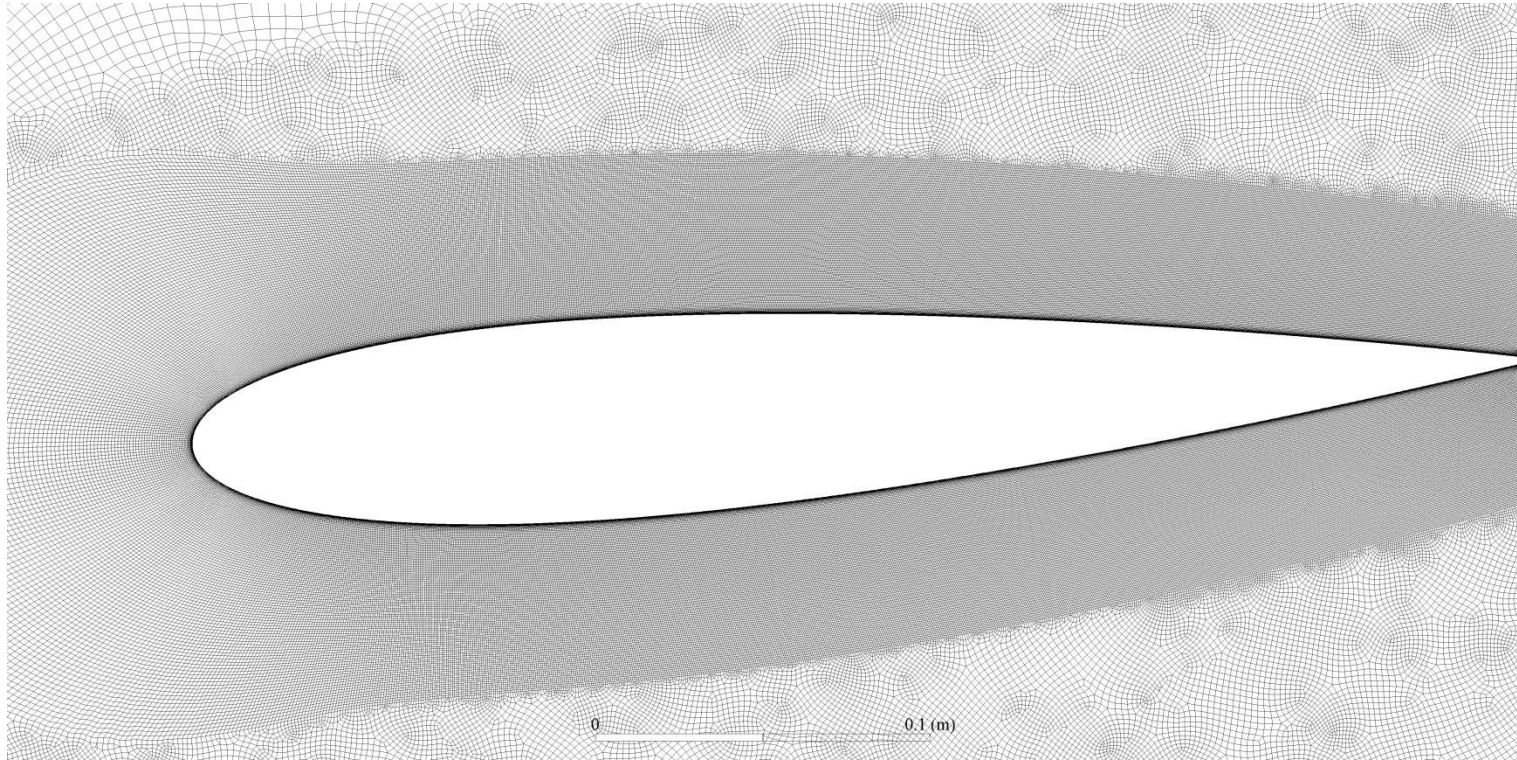


Figure 16. Grid distribution around NACA 0015



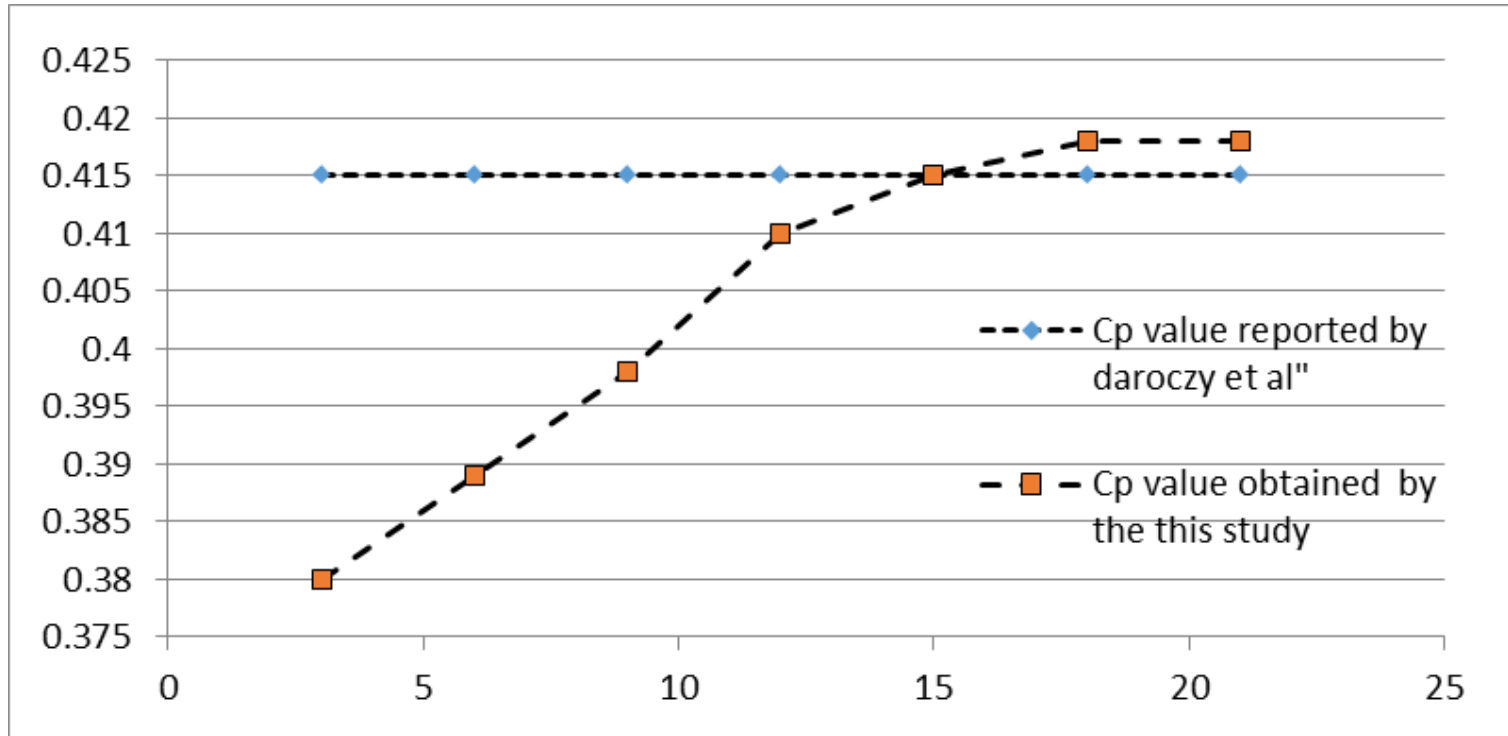


Figure 17. Mesh independence study



Numerical setup

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Simulation assumptions:

- k- ω SST model.
- SIMPLE algorithm .
- Utilizing a second-order upwind approach.
- The outlet boundary is represented as a pressure outlet.
- the surface of the airfoil is described as a non slip wall.
- the inlet boundary is represented as a velocity inlet.

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Azimuthal increments served as the foundation for the transient simulation's time step. To replicate the five λ of interest, five unique rotating speeds ω (rad/s) were considered: 0.2 (to examine the starting torque), 1.25, 1.6, 2 and 2.5. , the time step in the present investigation was defined as

$$\Delta t = 2\pi \Delta\theta / (360\omega) \quad (13)$$

Where Δt time is step size and $\Delta\theta$ is azimuthal increment. (0.5°) was utilized as the azimuthal increment at each time step in agreement with comparable numerical work published in [11]. Equation (13) was used to compute the appropriate time steps depending on the values of ω and $\Delta\theta$.



λ	$\omega(\text{rad/s})$	Time step (s)
0.2	1.6	0.005454
1.25	10	0.0008727
1.6	12.8	0.0006817
2	16	0.0005454
2.5	20	0.0004363

Table 3. ω and Δt for different λ .

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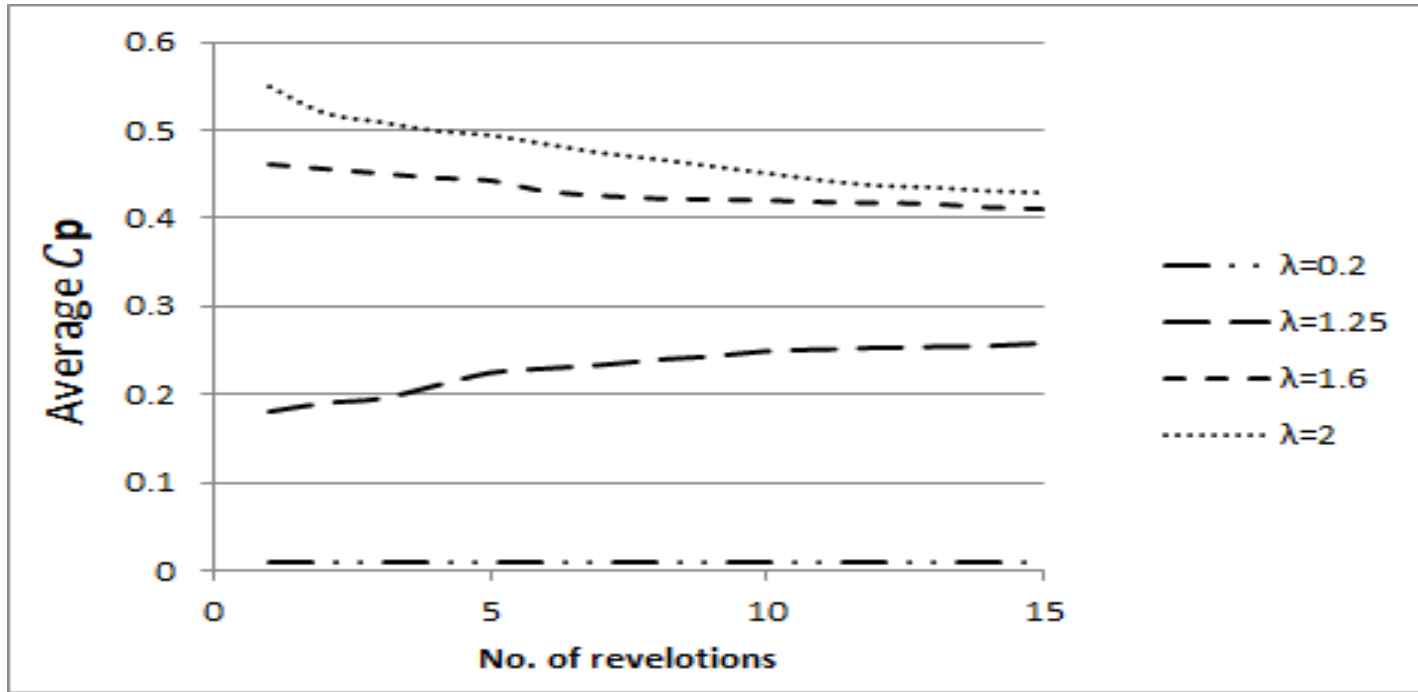


Figure 18. Number of revolutions until convergence of simulated Cp values at various λ .

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VERIFICATION AND VALIDATION

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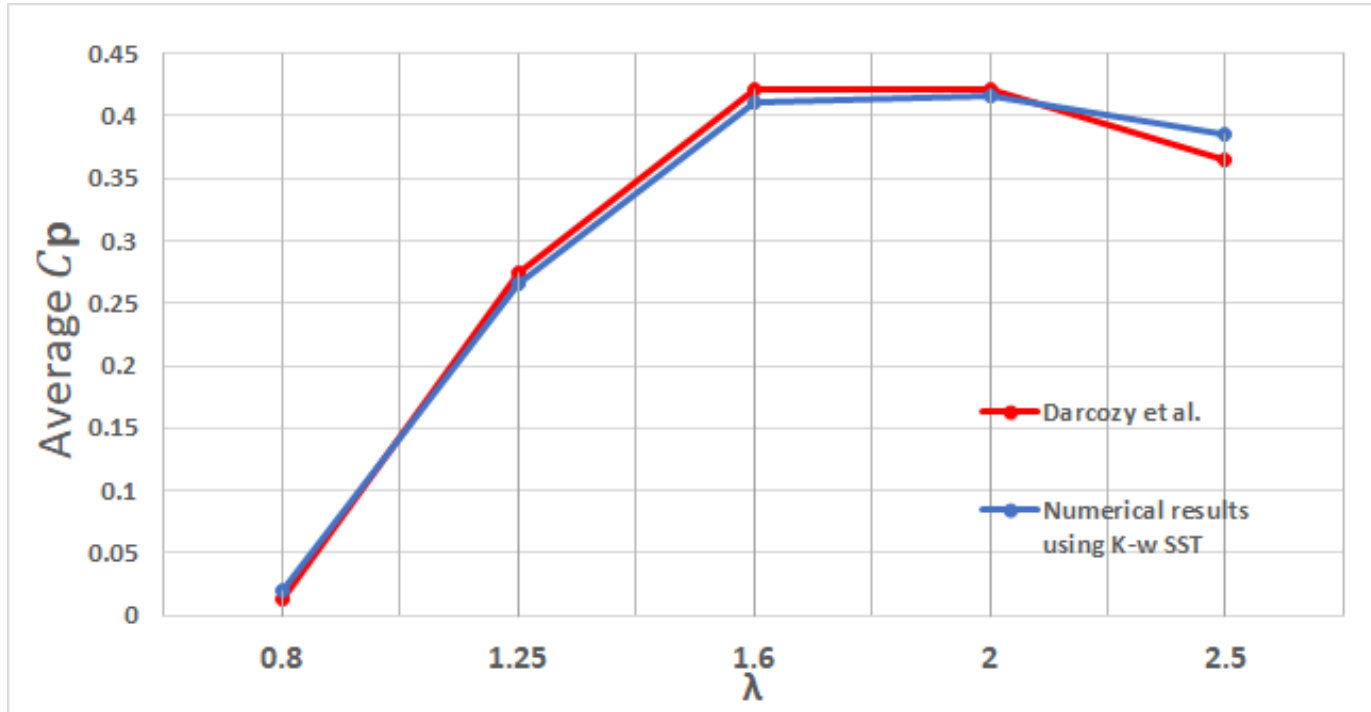


Figure 19. C_p comparison between the present study and Daróczy et al. at different λ

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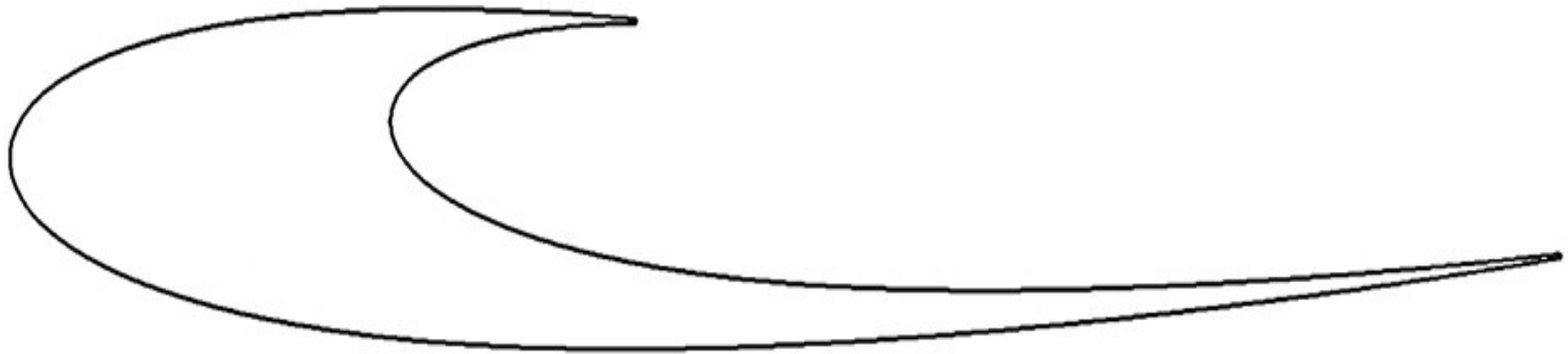




J-BLADE CONFIGURATION

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Figure 20.(a) J-blade airfoil design



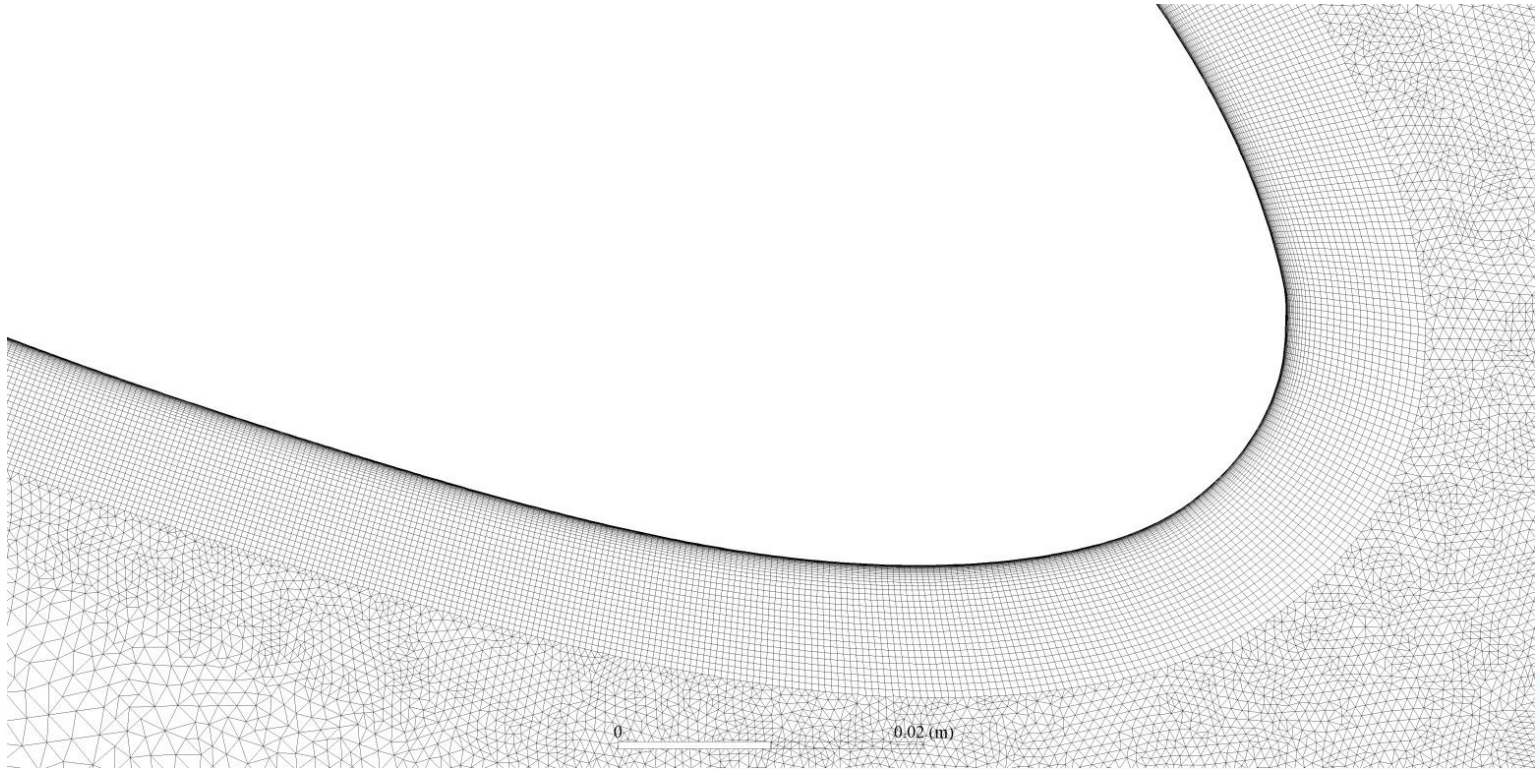


Figure 20. (b) the grid distribution

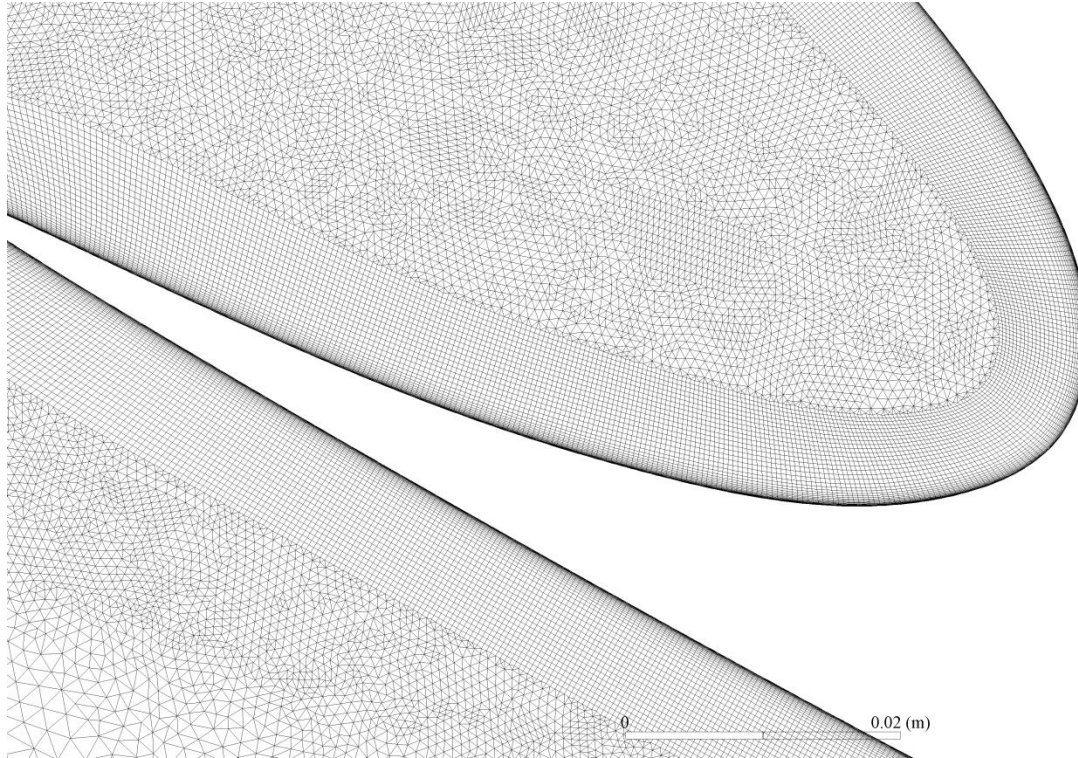


Figure 20. (b) the grid distribution

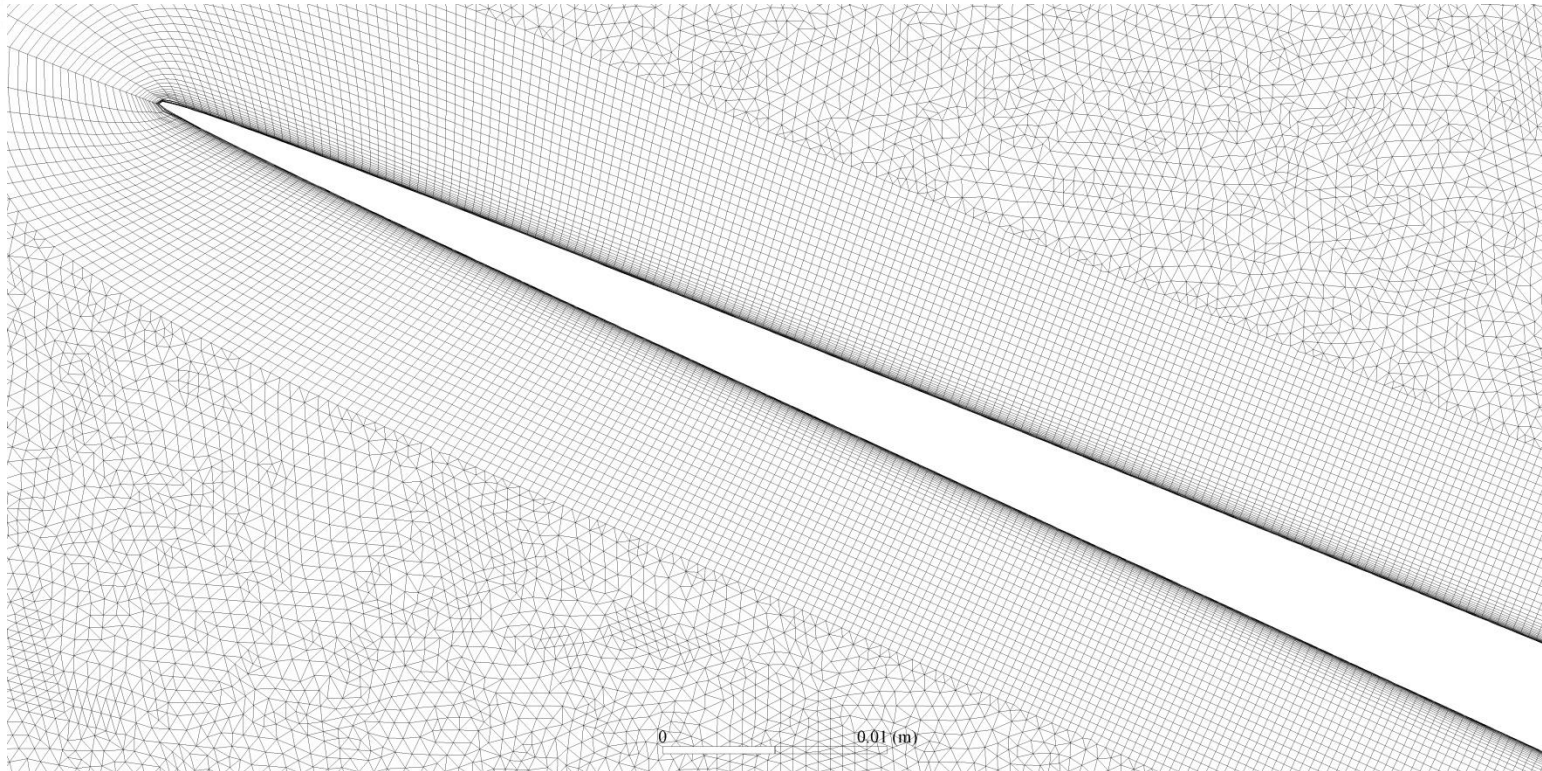


Figure 20. (b) the grid distribution





RESULTS AND DISCUSSION

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Standard operating conditions

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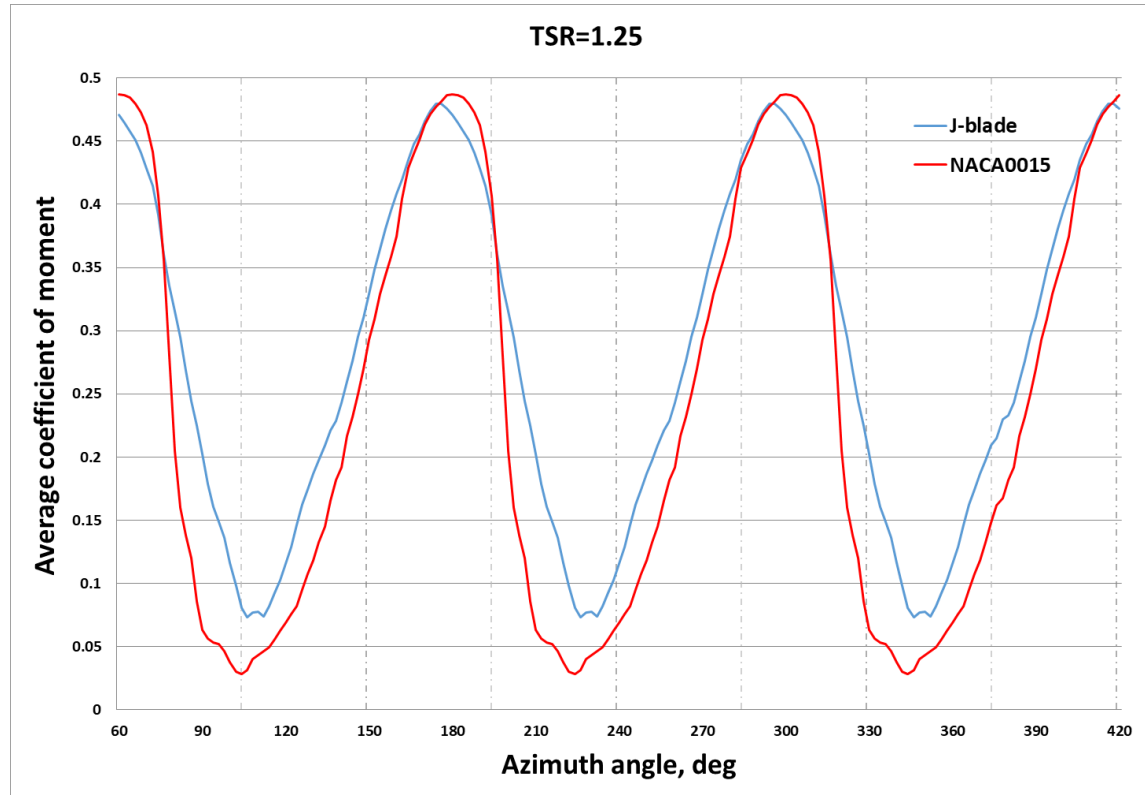


Figure 23. Cm comparison between NACA0015 (red) and J-blade (blue) at $\lambda = 1.25$ (a)

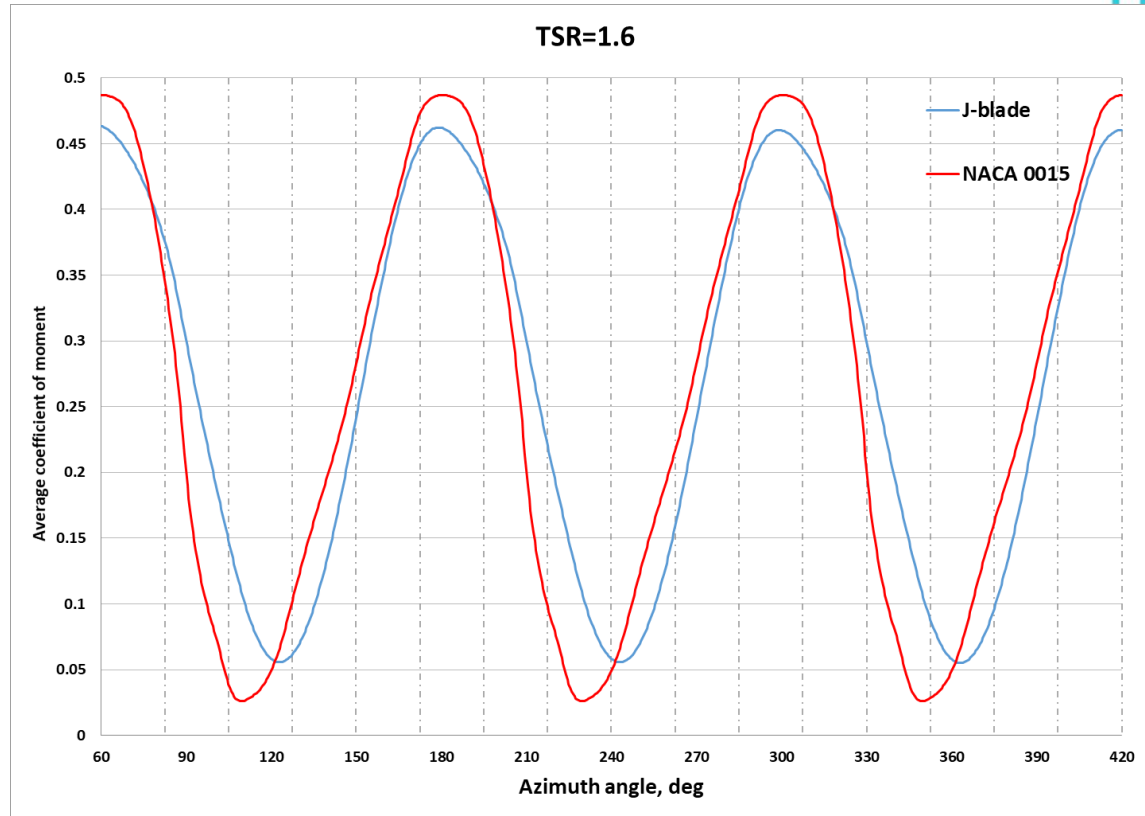


Figure 24. Cm comparison between NACA0015 (red) and J-blade (blue) at $\lambda = 1.6$ (b)

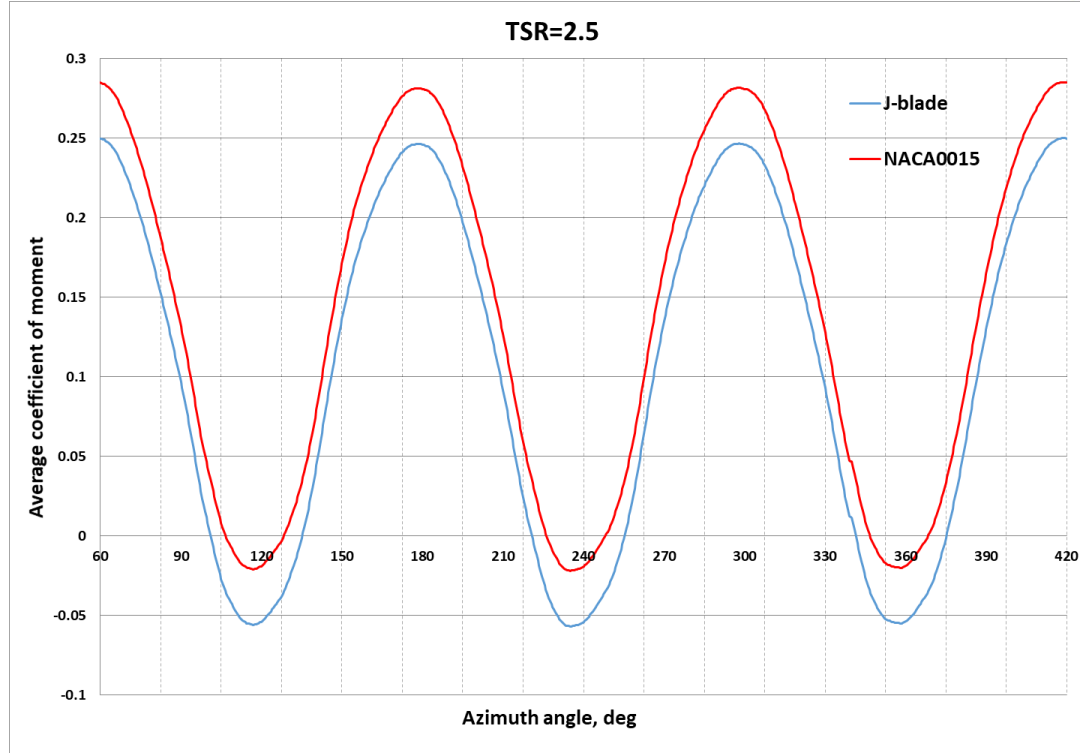


Figure 25. Cm comparison between NACA0015 (red) and J-blade (blue) at $\lambda = 2.5(c)$

- The torque values of the J-blade airfoil and the NACA0015 airfoil are comparable. For the majority of the operating zone, the J-blade airfoil outperformed the NACA0015 airfoil in terms of performance.
- The J-blade airfoil generated torque equivalent to the NACA0015 airfoil at λ of 1.25 and 1.6, but it was more uniform, which added benefits in terms of energy generation and mechanical stresses. Nevertheless, the J-blade airfoil's performance began to lag behind that of the NACA0015 airfoil at λ of 2.5.

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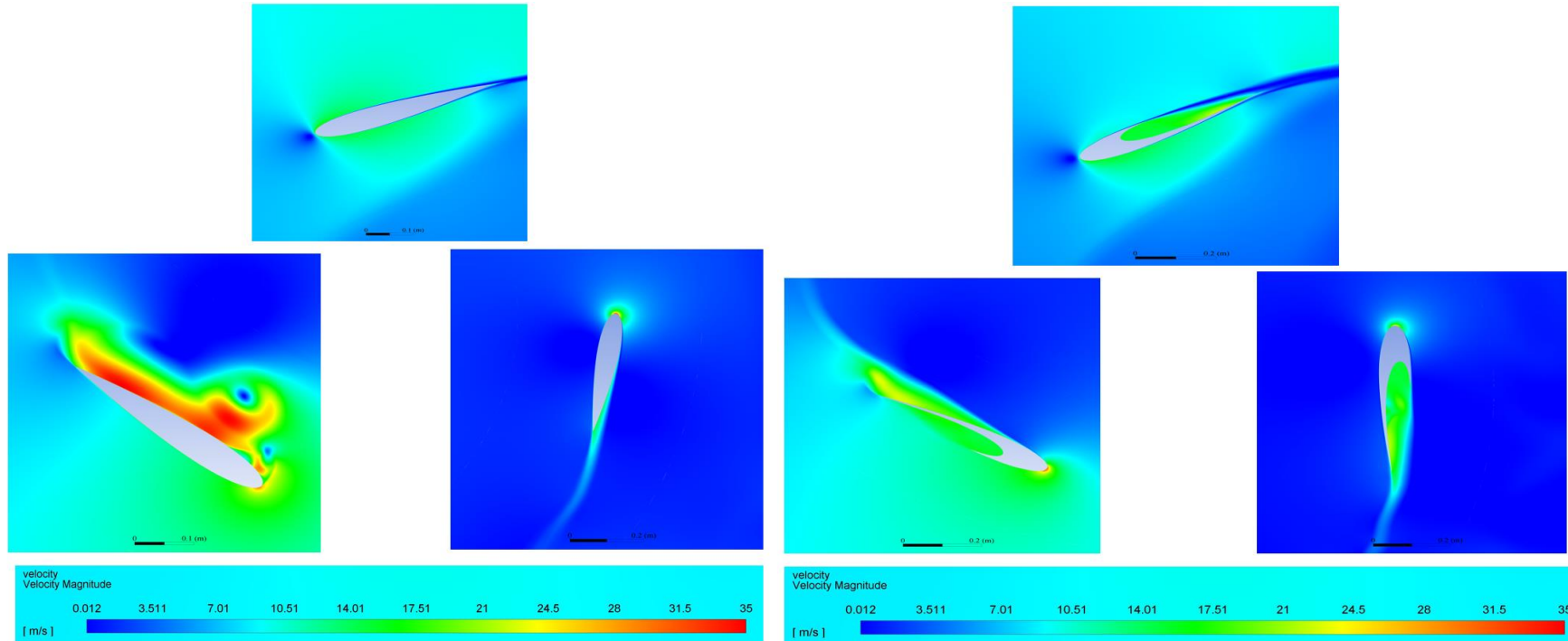
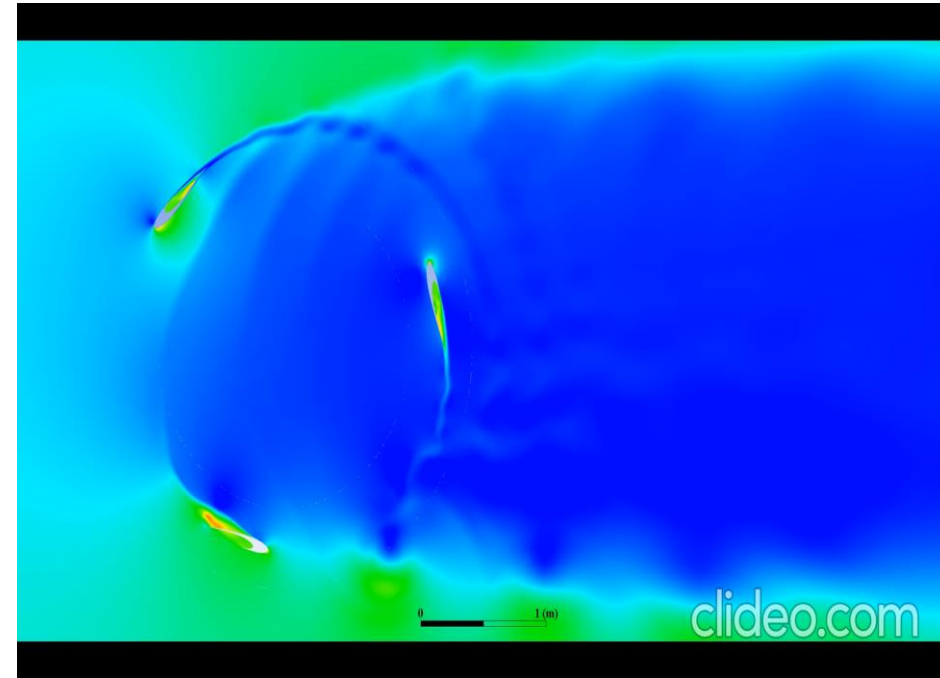
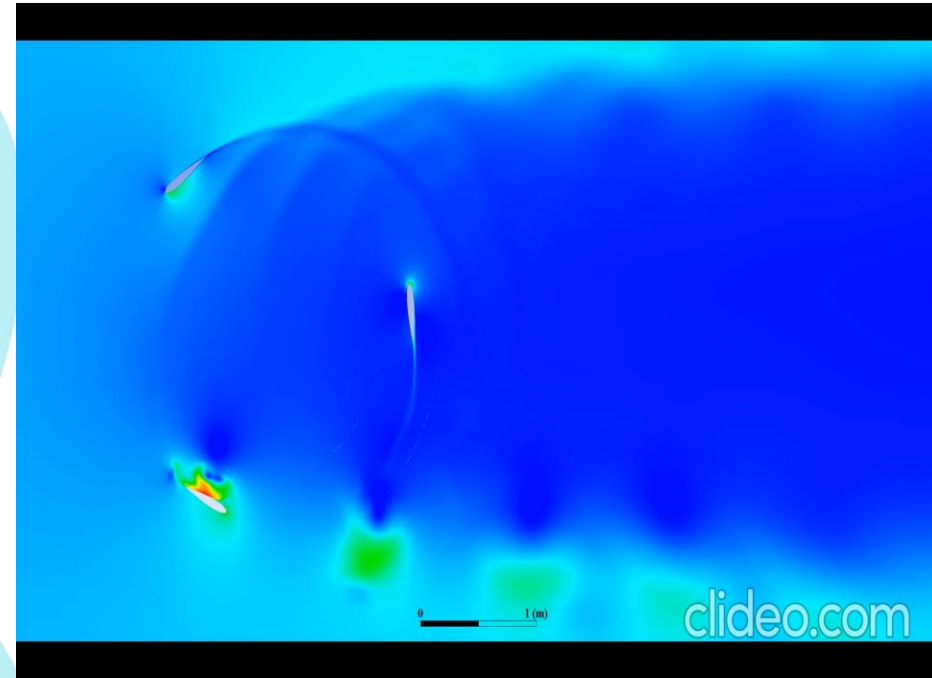


Figure 26. Contours of the velocity magnitude at $\lambda = 1.6$ around the airfoil for NACA0015 (left) and J-blade (right)





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Figure 26. Contours of the velocity magnitude at $\lambda = 1.6$ around the airfoil for NACA0015 (left) and J-blade (right)



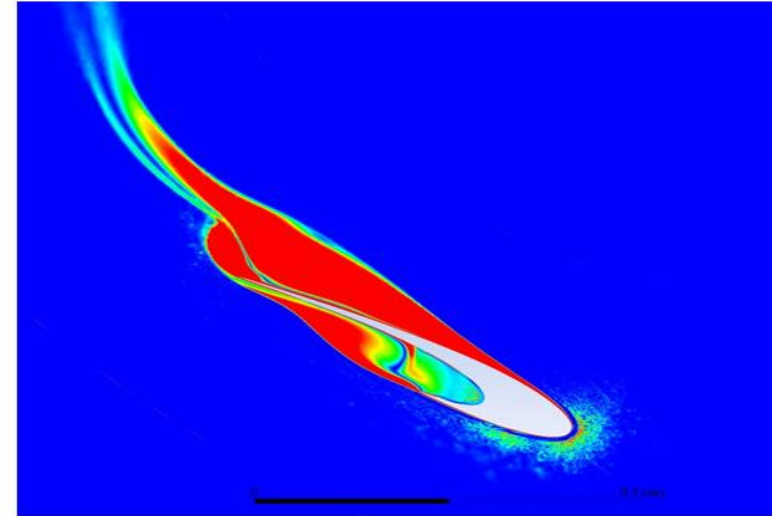
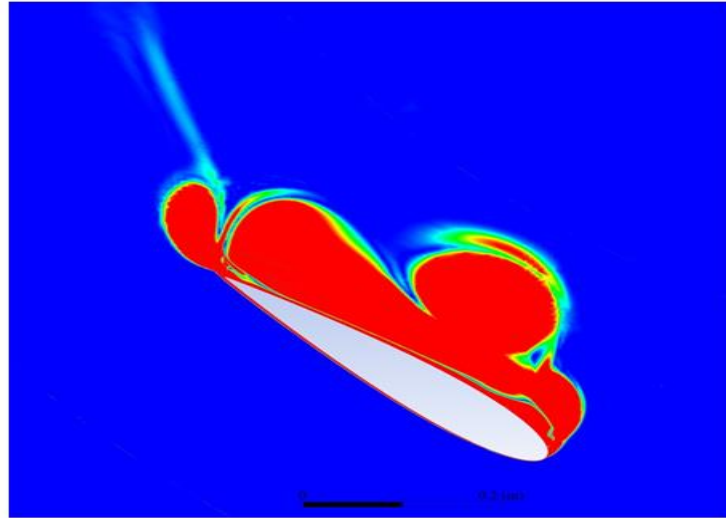


Figure 27. Contours of Gromeka acceleration vector magnitude in leeward region At $\lambda = 1.6$ Around blade for NACA0015(left) and J-blade (right)

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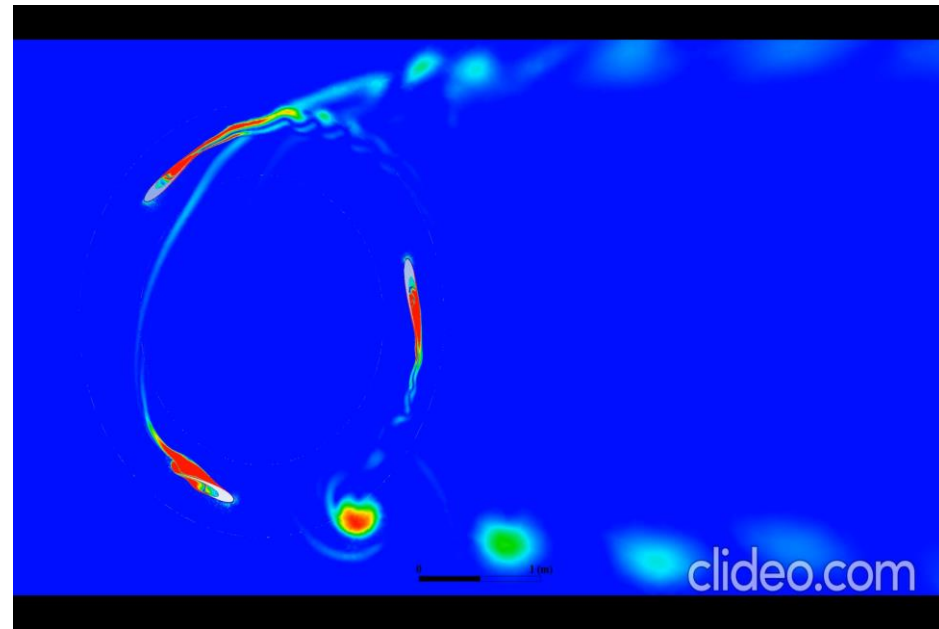
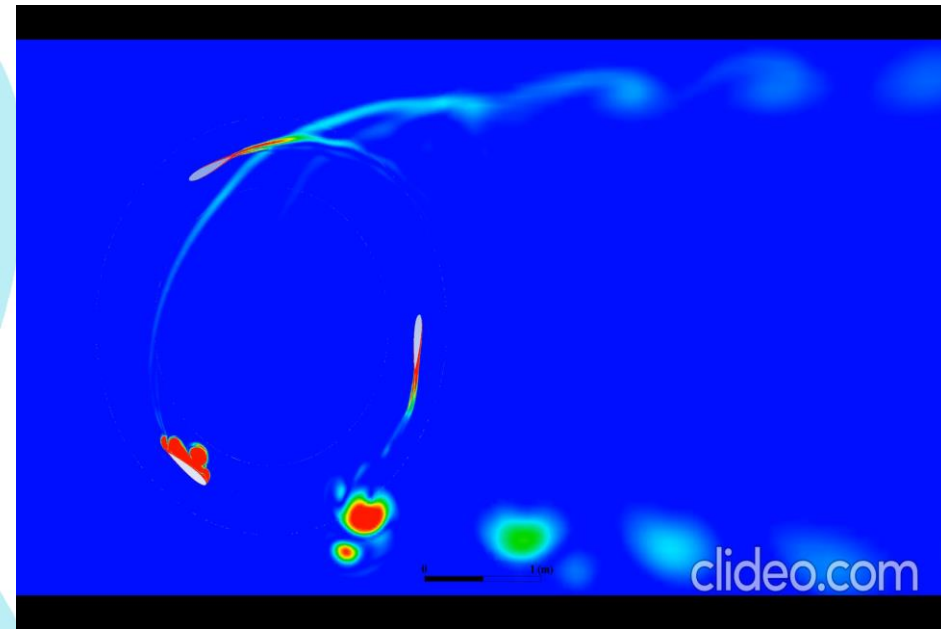


Figure 27. Contours of Gromeka acceleration vector magnitude in leeward region At $\lambda = 1.6$ Around blade for NACA0015(left) and J-blade (right)

TOPICS



- On the pressure side, the J-blade airfoil displays vortexes, yet the flow characteristics are not significantly affected by this cut. Figure 24 illustrates how the slight increase in drag causes just a slight reduction in torque in that area.
- With regard to the contours in the leeward zone, both airfoils exhibit significant flow separation. It is noteworthy, however, that the J-blade design exhibits the lowest vortex production, as shown in Figures 26, 27.

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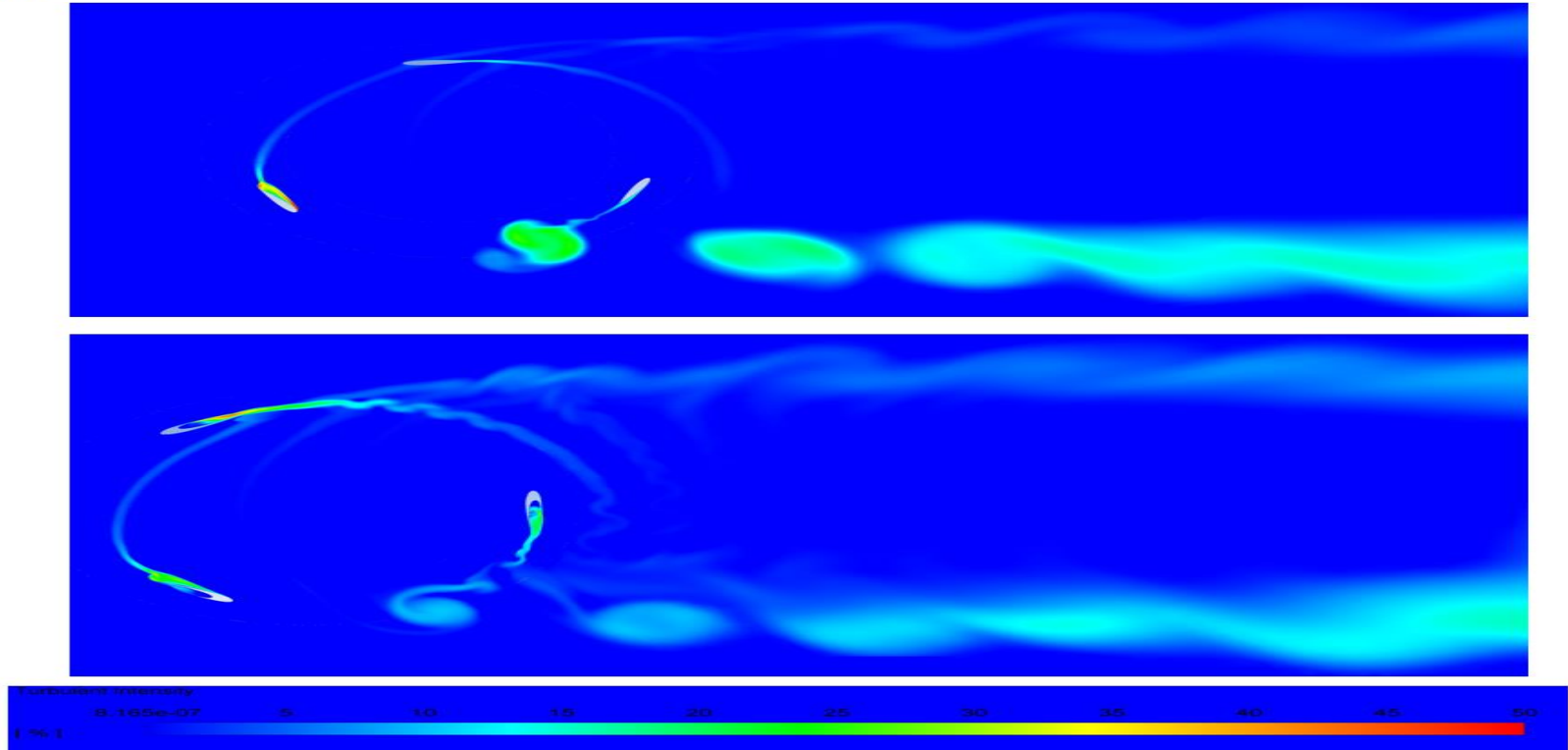
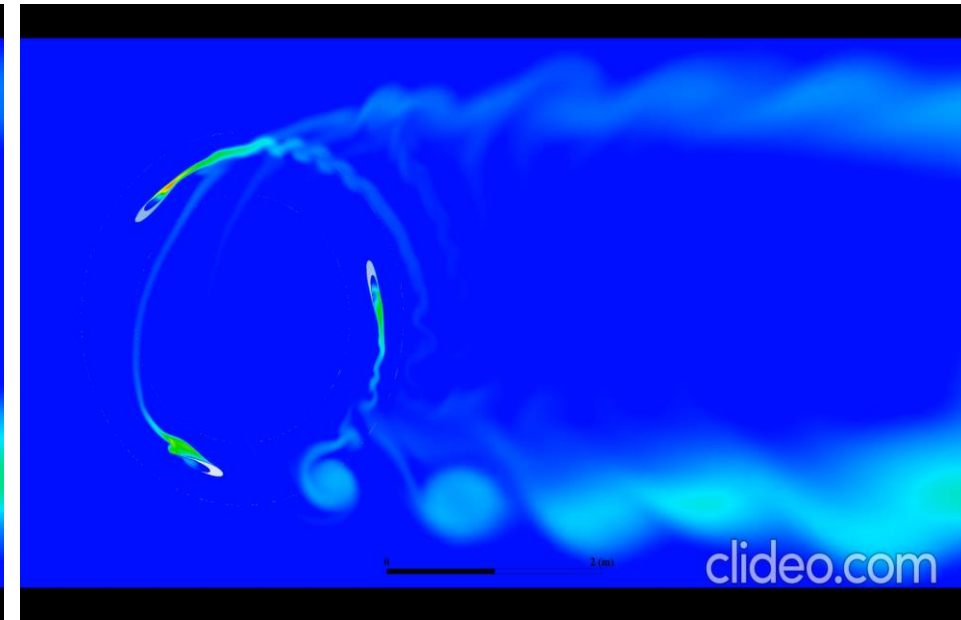
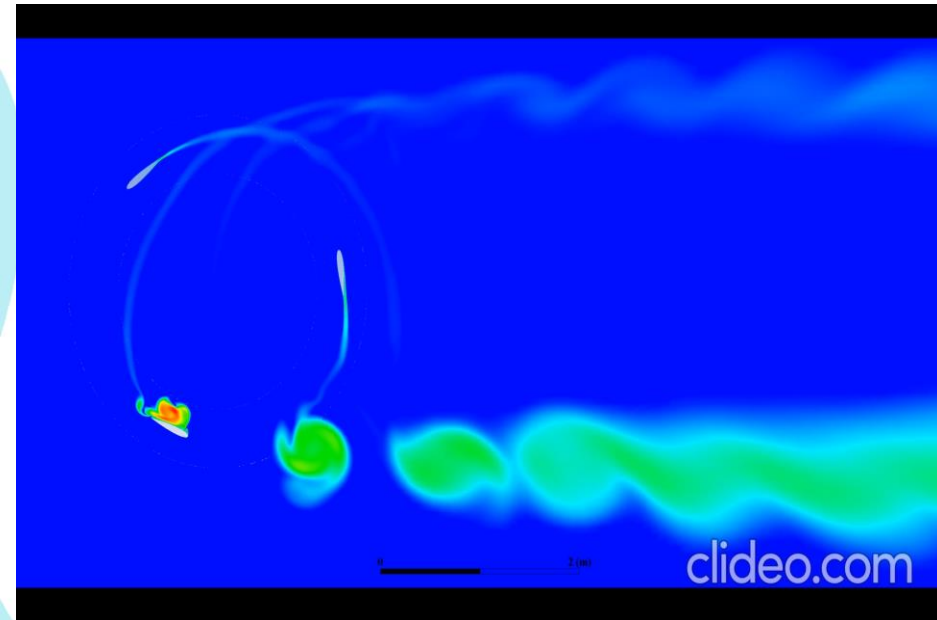


Figure 28. Contours of turbulence intensity $\lambda = 1.6$ around the turbine for NACA0015 (upper) and J-blade (down)





topics

Figure 28. Contours of turbulence intensity $\lambda = 1.6$ around the turbine for NACA0015 (upper) and J-blade (down)



- Thus, with $\lambda = 1.6$, Figure 28 displays examples of the turbulence fields for this airfoil. The vortex shedding in the leeward zone is a substantial influence detected in the velocity field.
- The Gromeka acceleration vector contour charts given in Figure 27 indicate how the J-blade airfoil greatly lowers the creation of vortices and shedding during passage in the leeward zone.
- In Figure 28, the contour plots of turbulence intensity are provided to offer a more complete study of the effect of this vortex production.



- Thus, combining both factors results in a uniform wake behind the turbine for the J-blade airfoil.
- Because turbulence from one turbine impacts the energy production of the next, we may use this consistency by deploying more and more wind turbines in the same geographical location, resulting in higher energy yields and cost savings.
- The instability in the wake raises the mechanical stresses and lowers the turbine's energy production.

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Starting Torque

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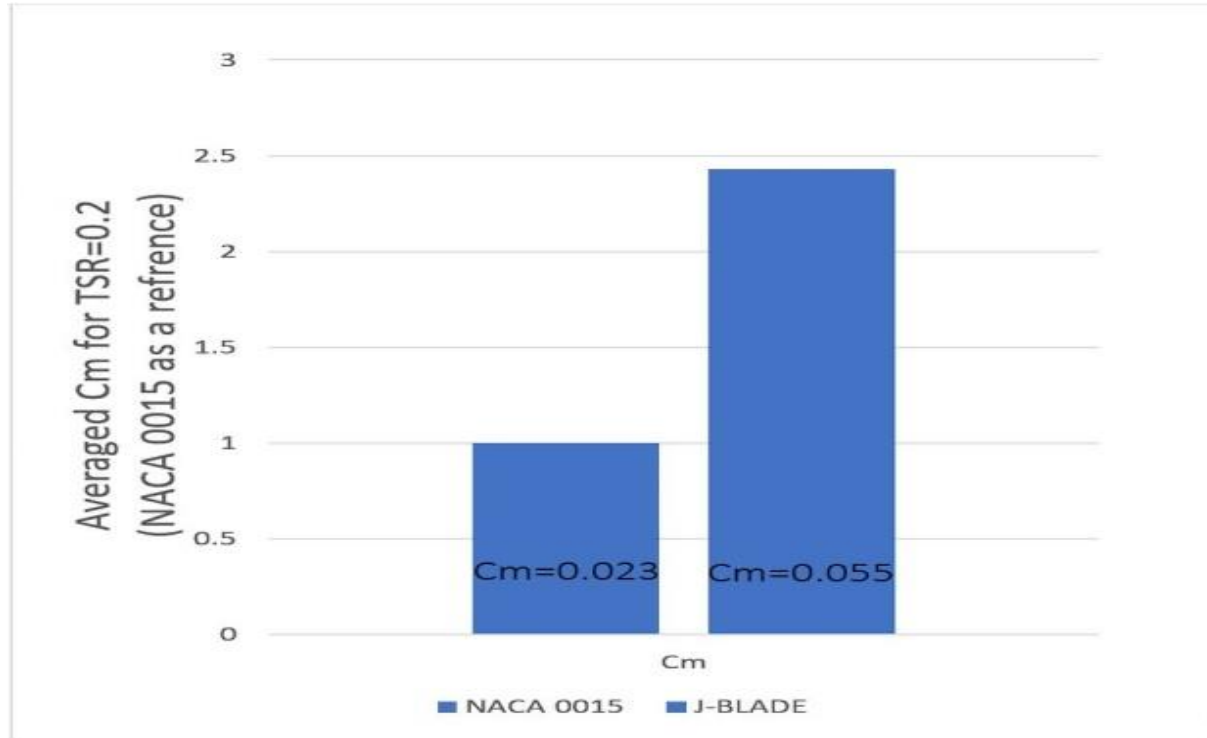


Figure 20. Averaged C_m for $\lambda=0.2$ of NACA0015 (left) and J-blade (Right)





- The simulated airfoil reveals that the J-blade airfoil has a **142%** increase in starting torque compared to the NACA0015 airfoil, as shown in figure 20. This rise in initial torque originates from the airfoil's cut, which causes additional drag going through the leeward zone.
- The J-blade consequently acts as a consequence of simultaneous lift and drag forces. In addition, the J-blade's notch provides drag force that helps the blades rotate more swiftly. Conversely, it enhances rotational efficiency by creating greater torque by harnessing the same wind more often.

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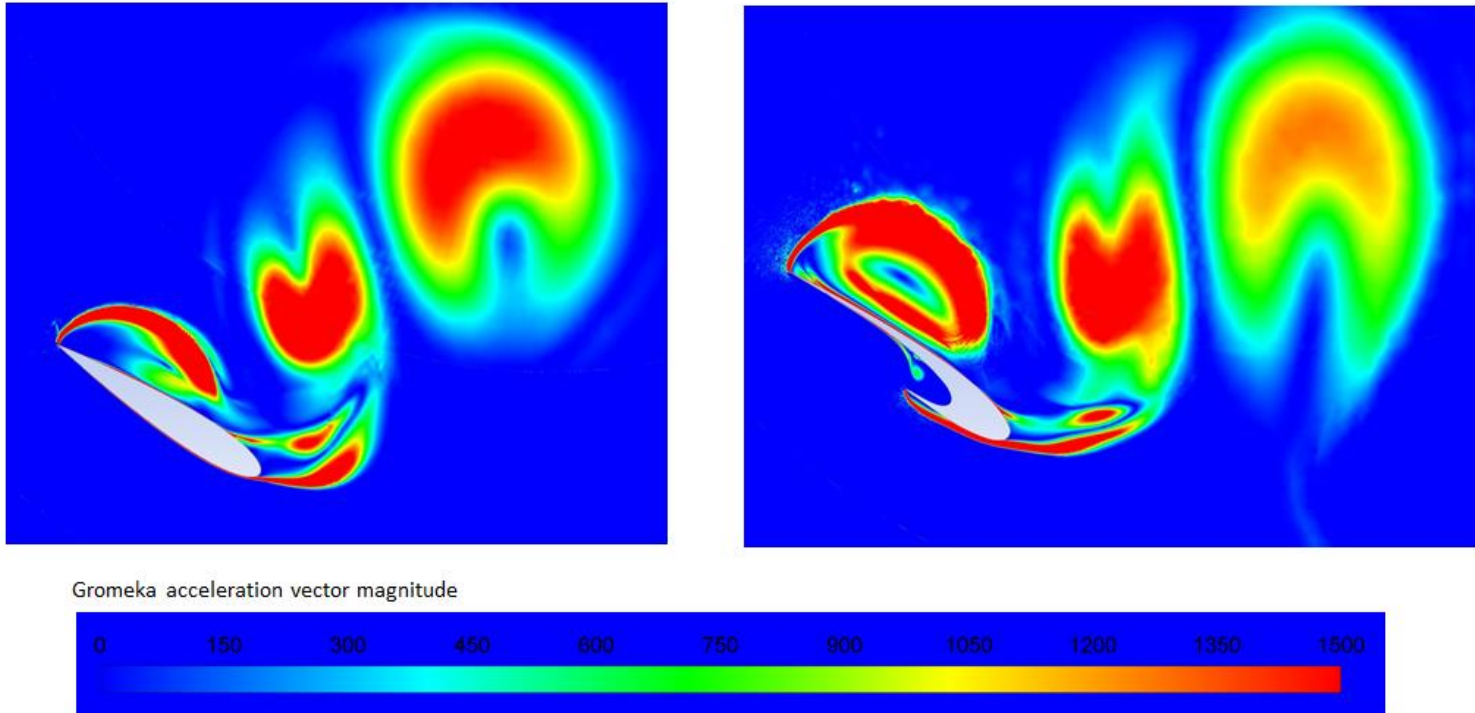


Figure 21. Contours of the Gromeka acceleration vector in leeward region at $\lambda = 0.2$ around the airfoil for NACA0015 (left) and J-blade (right)

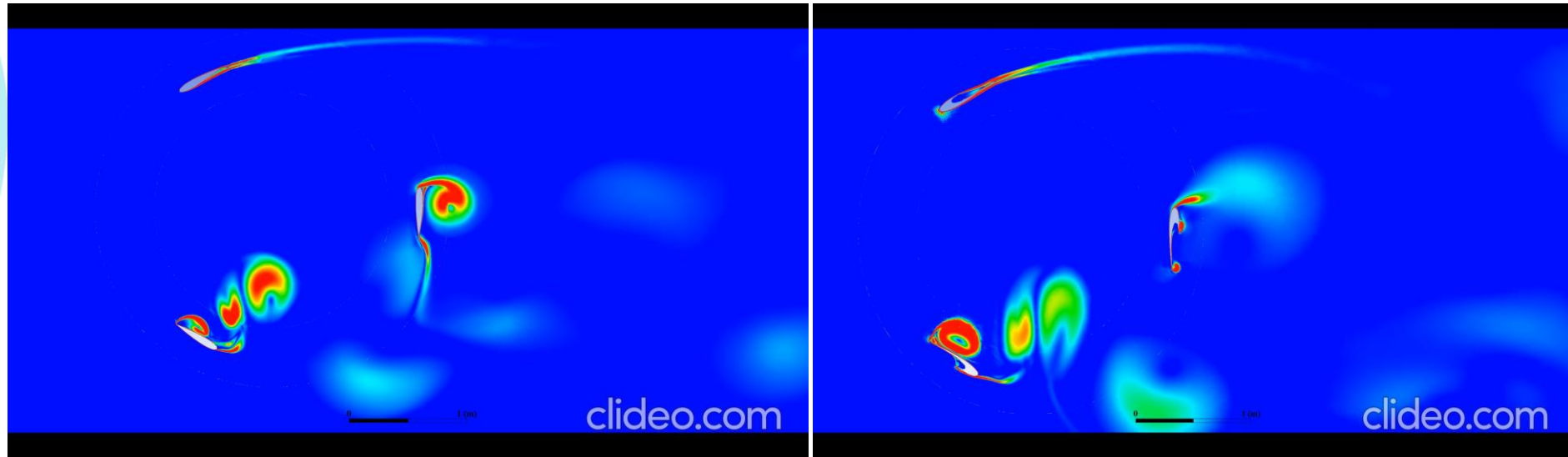


Figure 21. Contours of the Gromeka acceleration vector in leeward region at $\lambda = 0.2$ around the airfoil for NACA0015 (left) and J-blade (right)

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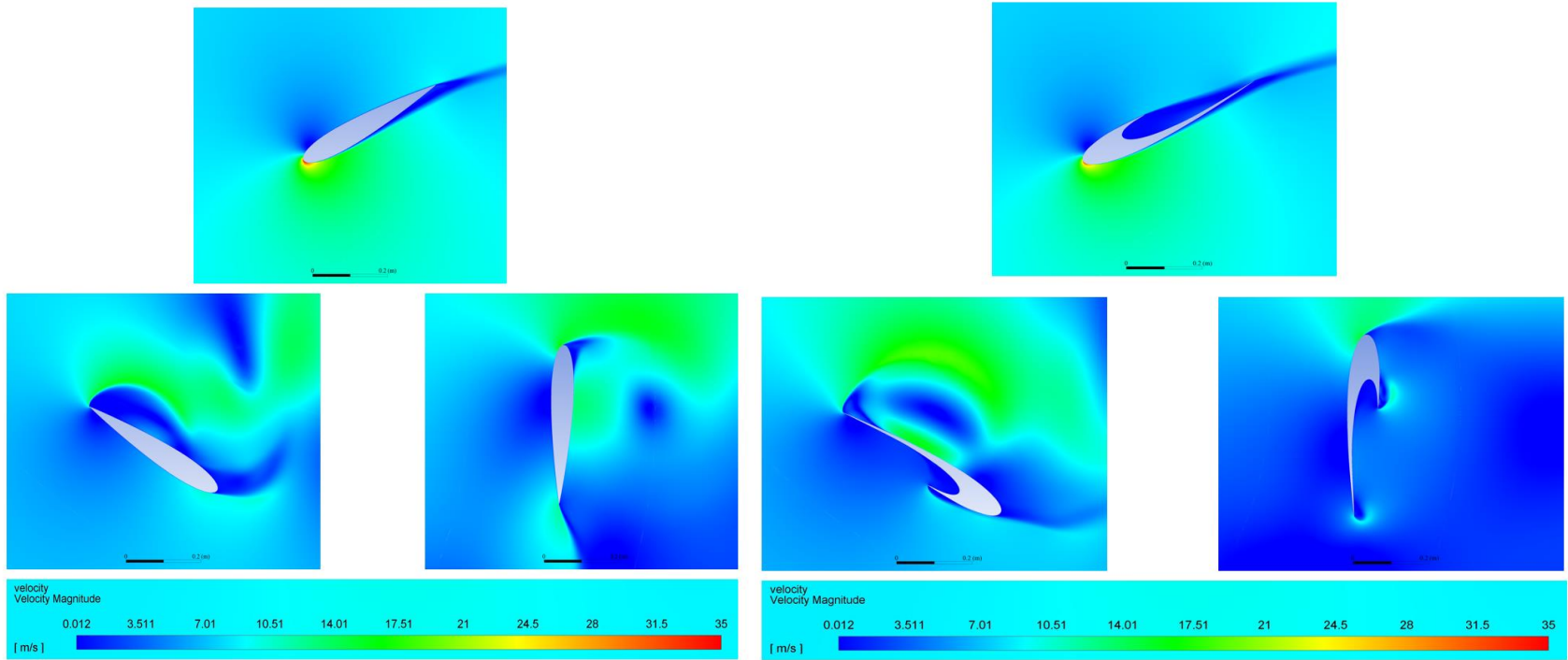
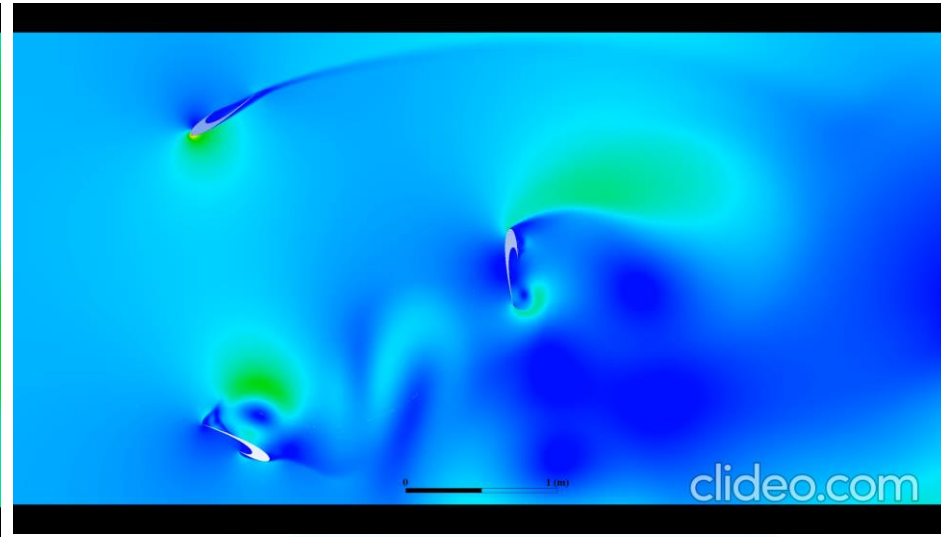
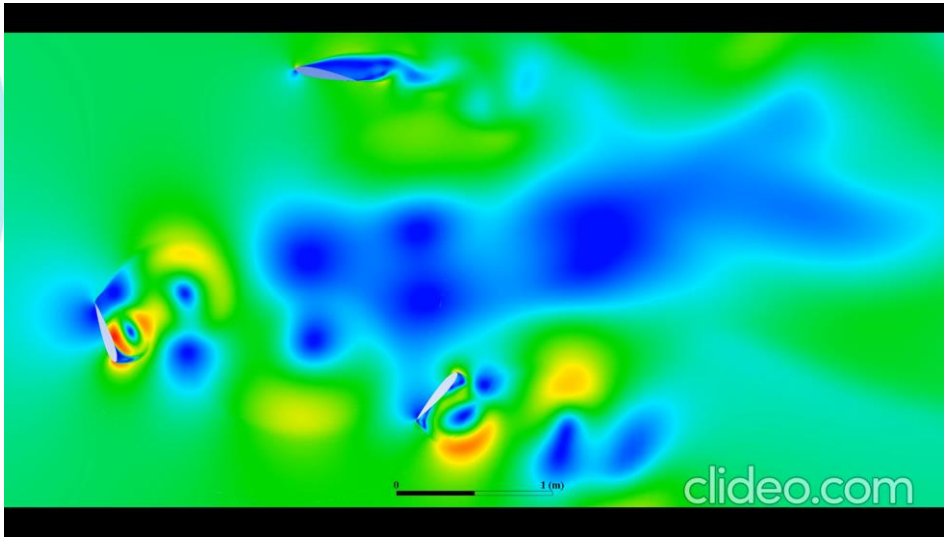


Figure 22. Contours of the velocity magnitude at $\lambda = 0.2$ around the airfoil for NACA0015 (left) and J-blade (right)





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Figure 22. Contours of the velocity magnitude at $\lambda = 0.2$ around the airfoil for NACA0015 (left) and J-blade (right)







Conclusions:

This research introduces and investigates the performance of an enhanced J-shaped blade for the Darrieus turbine, specifically tailored to urban environments. Using the URANS governing equations, a turbulent, incompressible, two-dimensional flow model was developed, focusing on both experimental validation and numerical models. The study systematically compared the proposed J-shaped blade with traditional NACA0015 blades, reproducing the Darrieus turbine to validate the numerical model in the range of low λ ($\lambda = 1.5$). The findings reveal that:

1. The J-shaped blade not only preserves power generation at the maximum efficiency point but also enhances uniformity, offering advantages in terms of fatigue stresses. This characteristic enables a more efficient placement of turbines in wind farms within the same land area.





2. Crucially, the J-shaped blade demonstrated a remarkable improvement in starting torque, achieving a torque that is 142% larger than that of the NACA0015 airfoil. This enhancement positions the turbine as a more viable and efficient solution for urban settings, addressing the challenges associated with initial rotation in low-wind conditions.

In summary, the integration of the proposed J-shaped blade not only maintains the overall performance of the Darrius turbine but also brings substantial advancements in starting torque, making it a promising innovation for urban wind energy applications. The research offers significant perspectives for the development and enhancement of Vertical-axis wind turbines. Paving the way for more sustainable and effective energy solutions in diverse environmental contexts.

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